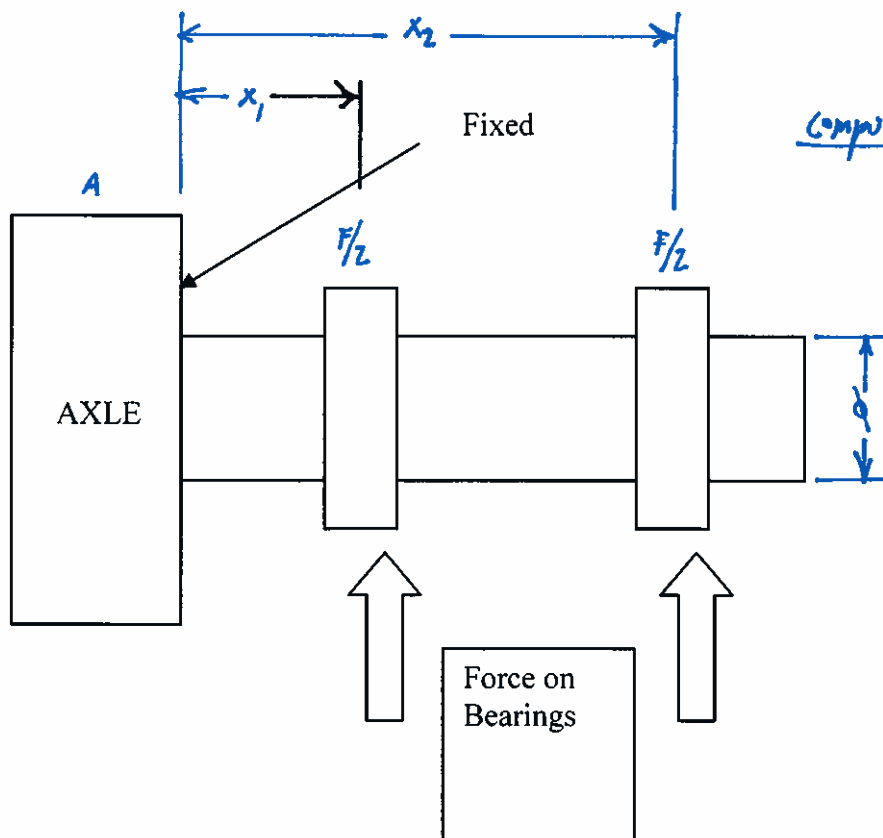
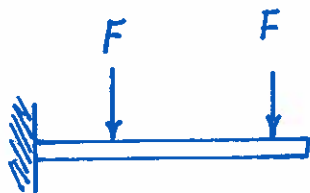




compute stress and size the shaft



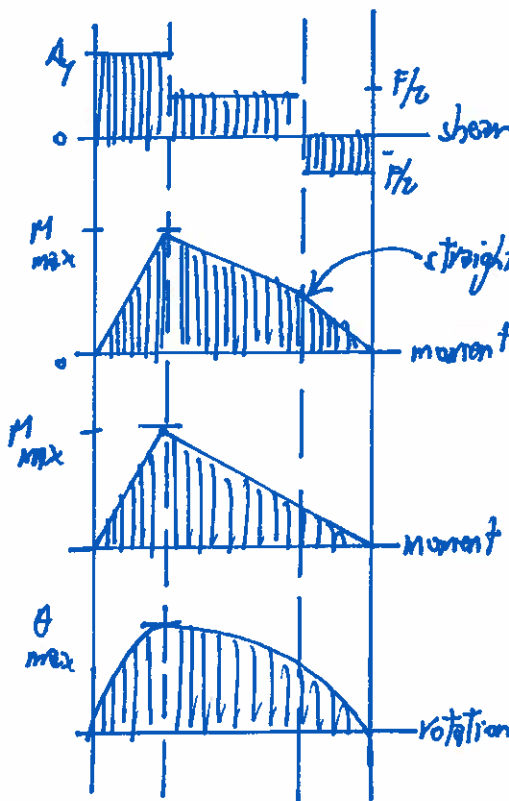
$$\uparrow \sum F_y = 0 = A_y - \frac{F}{2} - \frac{F}{2} \Rightarrow \underline{A_y = F}$$

$$\circlearrowleft \sum M_A = 0 = M_A - x_1 \left(\frac{F}{2} \right) - x_2 \left(\frac{F}{2} \right) \Rightarrow M_A = (x_2 - x_1) \frac{F}{2}$$

$$\sigma = \frac{Mc}{I} \quad \text{where} \quad I = \frac{1}{64} \pi \phi^4 \quad \text{and} \quad c = \frac{\phi}{2}$$

$$\sigma = \frac{(x_2 - x_1) \frac{F}{2} \cdot \frac{\phi}{2}}{\frac{\pi \phi^4}{64}} = \frac{16(x_2 - x_1)F}{\pi \phi^3}$$

$$\therefore \sigma = \frac{16(x_2 - x_1)F}{\pi \phi^3} \quad \text{stress at axle support}$$



• Compare this to material strength, σ_y which is a known.

• factor of safety: $FOS = \frac{\sigma_y}{\sigma}$

• you can set FOS to compute σ and given σ_y , size the shaft.

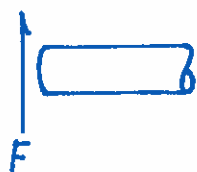
$$\frac{\sigma_y}{FOS} = \frac{16(x_2 - x_1)F}{\pi \phi^3} \Rightarrow \phi = \sqrt[3]{\frac{16(x_2 - x_1)F}{\pi \sigma_y} \cdot FOS}$$

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shear

$$\tau = \frac{F}{A} = \frac{F}{\frac{\pi}{4} \phi^2} = \frac{4F}{\pi \phi^2}$$

$$\therefore \underline{\tau = \frac{4F}{\pi \phi^2}}$$



• simple shear acts to cut the shaft at the support, one plane only!

Mohr's Circle

$$\sigma_{\min/\max} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

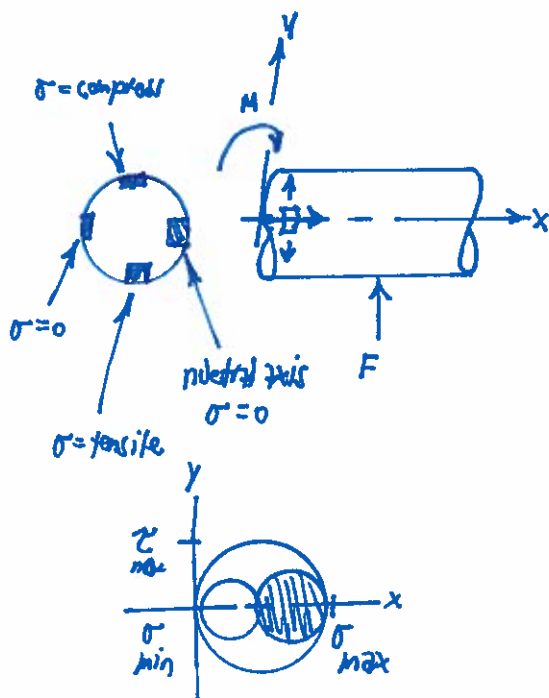
$$\tan 2\theta = \frac{\sigma_x - \sigma_y}{2\tau_{xy}}$$

$$\sigma_x = 0, \sigma_y = \frac{16(x_c - x_1)F}{\pi \phi^3}, \tau_{xy} = \frac{4F}{\pi \phi^2} \text{ the input values.}$$

$$\sigma_{\min/\max} = \frac{16(x_c - x_1)F}{2\phi^3\pi} \pm \sqrt{\left(\frac{16(x_c - x_1)F}{\pi \phi^3}\right)^2 + \left(\frac{4F}{\pi \phi^2}\right)^2} \quad \text{equation A}$$

$$2\theta = \tan^{-1} \left(\frac{-16(x_c - x_1)F}{\pi \phi^3} \cdot \frac{\pi \phi^2}{2 \cdot 4F} \right)$$

$$\tan \theta = \frac{1}{2} \tan^{-1} \left(\frac{-2(x_c - x_1)F}{F \cdot \phi} \right) \quad \text{equation B}$$



- so you can get $\sigma_{\min}$ as the "-" and $\sigma_{\max}$ as the "+" from eqn A
- the deformation of wall element is $\tan 2\theta$; gives an idea of shear role in the problem.
- at the neutral axis, stress is zero but shear is max/min.
- 90° off the neutral axis, the wall element experiences compression or tension; but the shear is zero. this is the principle stress.