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[ STUDENT > # Solve desertfox' example problem
[ STUDENT > # (Convert to metric)
[ STUDENT > # Find angle to determine if bearing reaction moment is
[ STUDENT > significant
[ STUDENT >
[ STUDENT > # Assume bearings and pulley have zero width
[ STUDENT >
[ STUDENT > # two sections: m = motor (between bearings); e =
[ STUDENT > extension
[ STUDENT > # Lm = distance between motor bearings
[ STUDENT > # Le = distance from inboard motor bearing to Pulley
[ STUDENT > center.
[ STUDENT > # x = distance coordinate.
[ STUDENT > # x=0 at OB brg. x=Lm at IB brg. x = Lm + Le at pulley
[ STUDENT >
[ STUDENT > # v = shear
[ STUDENT > # M = moment
[ STUDENT > # T = slope (T for theta)
[ STUDENT > # Y = displacement
[ STUDENT >
[ STUDENT > # Dirac = unit impulse (for concentrated load)
[ STUDENT > # Heaviside = unit step function
[ STUDENT >
[ STUDENT > restart; # initialization
[ STUDENT >
[ STUDENT > # Convert Inputs to metric
[ STUDENT > Dm:=evalf(4/40); # Diam of motor section of shaft (betw
[ STUDENT > brgs)


$$Dm := .1000000000$$


[ STUDENT > De:=evalf(2/40); # Diam of extension section of shaft
[ STUDENT > (betw brgs)


$$De := .0500000000$$


[ STUDENT > Lm:=evalf(10/40); # Diam of motor section of shaft (betw
[ STUDENT > brgs)


$$Lm := .2500000000$$


[ STUDENT > Le:=evalf(5/40); # Length of extension section of shaft
[ STUDENT > (betw brgs)


$$Le := .1250000000$$


[ STUDENT > E:=200E9; # Young's Modulus in Pascals


$$E := .200 \cdot 10^{12}$$


[ STUDENT > # Area Moment Of Inertia
[ STUDENT > I_m:=evalf(Pi*Dm^4/64); # motor section

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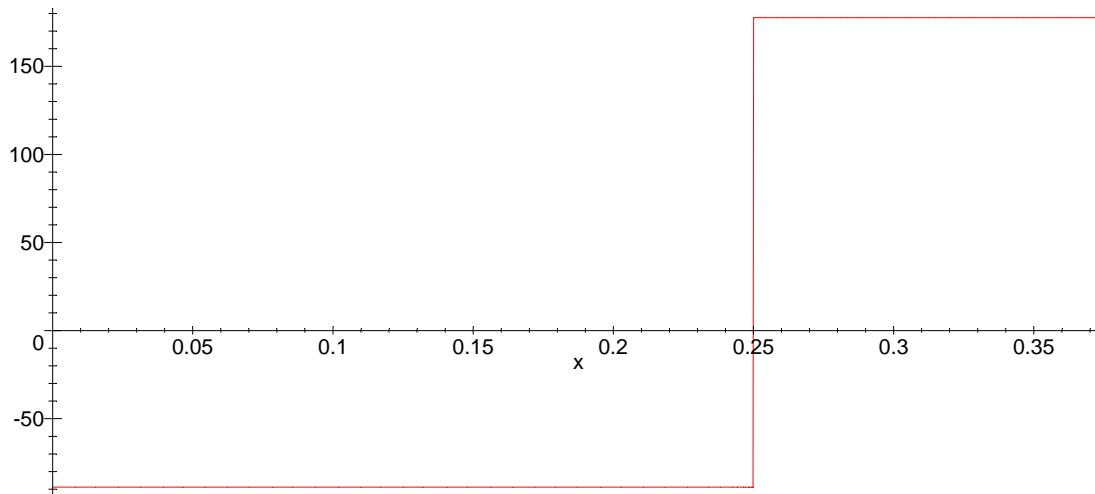
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I_m := .4908738522 10-5
STUDENT > I_e:=evalf(Pi*De^4/64); # extension section
I_e := .3067961576 10-6
STUDENT > Fp:=-40*4.44; # Pulley Force
Fp := -177.60
STUDENT > # Solve Reaction Forces from Force & Moment Balance
STUDENT > ForceBalanceEqn:=Fp+Rib+Rob=0;
ForceBalanceEqn := -177.60 + Rib + Rob = 0
STUDENT > MomentBalanceEqn:=Rib*Le+Rob*(Le+Lm)=0;
MomentBalanceEqn := .1250000000 Rib + .3750000000 Rob = 0
STUDENT > Rsolution:=solve({ForceBalanceEqn,MomentBalanceEqn},{Rib,R
ob});
Rsolution := {Rib = 266.4000000, Rob = -88.80000000}

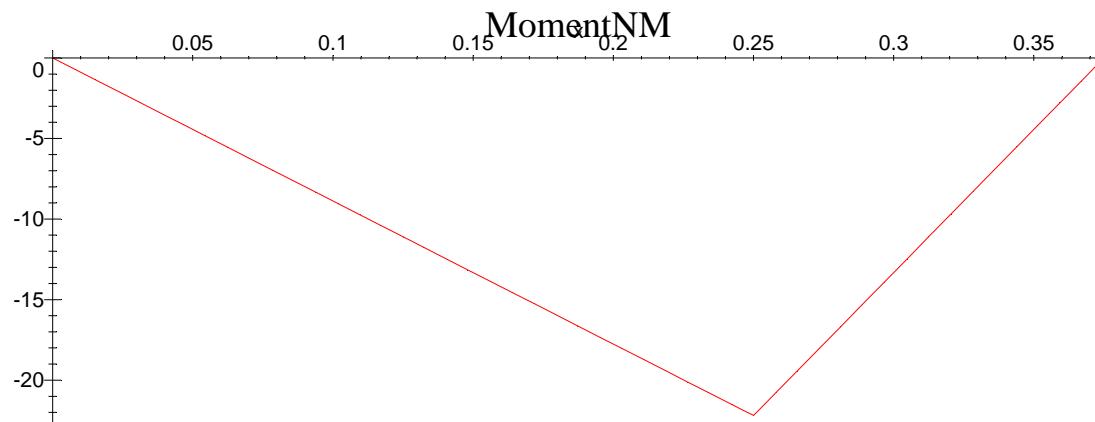
STUDENT > Rob:=subs(Rsolution,Rob); # Reaction force @ ob brg
Rob := -88.80000000
STUDENT > Rib:=subs(Rsolution,Rib); # Reaction force @ ib brg
Rib := 266.4000000
STUDENT > # Force profile is sum of applied force and reactions
STUDENT > Force(x):=Rob*Dirac(x-0)+Rib*Dirac(x-Lm)+Fp*Dirac(x-Lm-Le)
;
Force(x) := -88.80000000 Dirac(x) + 266.4000000 Dirac(x - .2500000000)
- 177.60 Dirac(x - .3750000000)
STUDENT > # note dirac is unit impulse
STUDENT >
STUDENT > # Solve shear as integral of force (no integration
constant since forces sum to 0)
STUDENT > V(x):=int(Force(x),x);
V(x) := -88.80000000 Heaviside(x) + 266.4000000 Heaviside(x - .2500000000)
- 177.6000000 Heaviside(x - .3750000000)
STUDENT >
STUDENT > plot(V(x),x=0..Lm+Le,title='ShearNewtons');

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## ShearNewtons



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STUDENT > # Solve Moment as integral of shear
STUDENT > M(x):=int(v(x),x)+CM;
M(x):=-88.80000000 Heaviside(x) x + 266.4000000 Heaviside(x -.2500000000) x
- 66.60000000 Heaviside(x -.2500000000) - 177.6000000 Heaviside(x -.3750000000) x
+ 66.60000000 Heaviside(x -.3750000000) + CM
STUDENT >
STUDENT > # Evaluate integration constant assuming free end moment
           is 0
STUDENT > CMSolution:=solve(subs(x=Le+Lm,M(x))=0,CM);
CMSolution := 0
STUDENT > M(x):=subs(CM=CMSolution,M(x));
M(x):=-88.80000000 Heaviside(x) x + 266.4000000 Heaviside(x -.2500000000) x
- 66.60000000 Heaviside(x -.2500000000) - 177.6000000 Heaviside(x -.3750000000) x
+ 66.60000000 Heaviside(x -.3750000000)
STUDENT > plot(M(x),x=0..Lm+Le,title='MomentNM');
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[ STUDENT > # Solve theta as 1/(E*I) * integral of Moment
[ STUDENT > # Divide into regions m (motor) and e (extension) because
    I differs
[ STUDENT > Tm(x):=1/(E*I_m)*int(M(x),x)+CTm;
Tm(x):=-.00004522546864 Heaviside(x)x^2 + .0001356764059 Heaviside(x-.2500000000)x^2
    + .8479775370 10^-5 Heaviside(x-.2500000000)
    - .00006783820296 Heaviside(x-.2500000000)x
    - .00009045093728 Heaviside(x-.3750000000)x^2
    - .00001271966305 Heaviside(x-.3750000000)
    + .00006783820296 Heaviside(x-.3750000000)x + CTm
[ STUDENT > Te(x):=1/(E*I_e)*int(M(x),x)+CTe;
Te(x):=-.0007236074979 Heaviside(x)x^2 + .002170822494 Heaviside(x-.2500000000)x^2
    + .0001356764059 Heaviside(x-.2500000000)
    - .001085411247 Heaviside(x-.2500000000)x
    - .001447214996 Heaviside(x-.3750000000)x^2
    - .0002035146088 Heaviside(x-.3750000000)
    + .001085411247 Heaviside(x-.3750000000)x + CTe
[ STUDENT >
[ STUDENT >
[ STUDENT > # Solve displacement as integral of slop
[ STUDENT > # Again solve in two regions
[ STUDENT > Ym(x):=int(Tm(x),x)+CYm;
Ym(x):=-.00001507515621 Heaviside(x)x^3 + .00004522546863 %2 x^3 - .7066479474 10^-6 %2
    + .8479775370 10^-5 %2 x - .00003391910148 %2 x^2 - .00003015031243 %1 x^3
    + .1589957880 10^-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x^2 + CTm x + CYm
    %1 := Heaviside(x-.3750000000)
    %2 := Heaviside(x-.2500000000)
[ STUDENT > Ye(x):=int(Tm(x),x)+CYe;
Ye(x):=-.001507515621 Heaviside(x)x^3 + .004522546863 %2 x^3 - .00007066479474 %2
    + .0008479775370 %2 x - .003391910148 %2 x^2 - .003015031243 %1 x^3 + .0001589957880 %1
    - .001271966305 %1 x + .003391910148 %1 x^2 + CTm x + CYe
    %1 := Heaviside(x-.3750000000)
    %2 := Heaviside(x-.2500000000)
Ye(x):=-.00001507515621 Heaviside(x)x^3 + .00004522546863 %2 x^3 - .7066479474 10^-6 %2
    + .8479775370 10^-5 %2 x - .00003391910148 %2 x^2 - .00003015031243 %1 x^3
    + .1589957880 10^-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x^2 + CTm x + CYe
    %1 := Heaviside(x-.3750000000)
    %2 := Heaviside(x-.2500000000)

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[ STUDENT >
[ STUDENT > # 4 Boundary conditions to solve the integration constants
[ STUDENT > BC1:=simplify(subs(x=0,Ym(x))=0); # Displacement # ob is 0
[ BC1 := CYm = 0
[ STUDENT > BC2:=simplify(subs(x=Lm,Ym(x))=0); # Displacement # ib is 0
[ BC2 := -.2355493158 10-6 + .2500000000 CTm + CYm = 0
[ STUDENT > BC3:=simplify(subs(x=Lm,Ye(x))=0); # Displacement # ib is 0
[ applies to both displacement solutions
[ BC3 := -.2355493158 10-6 + .2500000000 CTm + CYe = 0
[ STUDENT > BC4:=simplify(subs(x=Lm,Tm(x))=subs(x=Lm,Te(x))); # continuity of slope @ ib
[ BC4 := -.2826591790 10-5 + CTm = -.00004522546862 + CTe
[ STUDENT > Csolution:=solve({BC1,BC2,BC3,BC4},{CTm,CTe,CYm,CYe});
[ Csolution := { CYe = 0, CYm = 0, CTm = .9421972632 10-6, CTe = .00004334107409 }
[ STUDENT >
[ STUDENT > # Plug solution constants back into T and Y
[ STUDENT > Tm(x):=subs(Csolution,Tm(x));
Tm(x) := -.00004522546864 Heaviside(x) x2 + .0001356764059 Heaviside(x - .2500000000) x2
+ .8479775370 10-5 Heaviside(x - .2500000000)
- .00006783820296 Heaviside(x - .2500000000) x
- .00009045093728 Heaviside(x - .3750000000) x2
- .00001271966305 Heaviside(x - .3750000000)
+ .00006783820296 Heaviside(x - .3750000000) x + .9421972632 10-6
[ STUDENT > Te(x):=subs(Csolution,Te(x));
Te(x) := -.0007236074979 Heaviside(x) x2 + .002170822494 Heaviside(x - .2500000000) x2
+ .0001356764059 Heaviside(x - .2500000000)
- .001085411247 Heaviside(x - .2500000000) x
- .001447214996 Heaviside(x - .3750000000) x2
- .0002035146088 Heaviside(x - .3750000000)
+ .001085411247 Heaviside(x - .3750000000) x + .00004334107409
[ STUDENT > Ym(x):=subs(Csolution,Ym(x));
Ym(x) := -.00001507515621 Heaviside(x) x3 + .00004522546863 %2 x3 - .7066479474 10-6 %2
+ .8479775370 10-5 %2 x - .00003391910148 %2 x2 - .00003015031243 %1 x3
+ .1589957880 10-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x2
+ .9421972632 10-6 x
%1 := Heaviside(x - .3750000000)

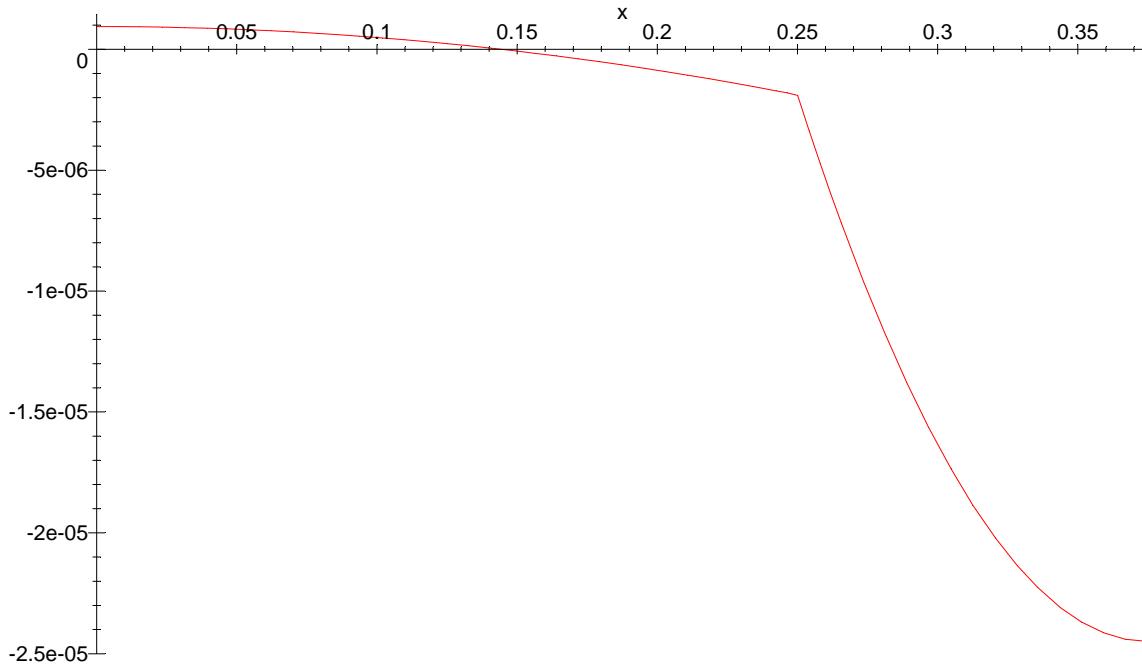
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%2 := Heaviside(x - .2500000000)
STUDENT > Ye(x):=subs(Csolution,Ye(x));
Ye(x) := -.00001507515621 Heaviside(x) x3 + .00004522546863 %2 x3 - .7066479474 10-6 %2
+ .8479775370 10-5 %2 x - .00003391910148 %2 x2 - .00003015031243 %1 x3
+ .1589957880 10-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x2
+ .9421972632 10-6 x
%1 := Heaviside(x - .3750000000)
%2 := Heaviside(x - .2500000000)
STUDENT > T(x):=piecewise(x>Lm,Tc(x),Tm(x));
T(x) := { -.0007236074979 Heaviside(x) x2 + .002170822494 %2 x2 + .0001356764059 %2
- .001085411247 %2 x - .001447214996 %1 x2 - .0002035146088 %1 + .001085411247 %1 x
+ .00004334107409 , .2500000000 < x
- .00004522546864 Heaviside(x) x2 + .0001356764059 %2 x2 + .8479775370 10-5 %2
- .00006783820296 %2 x - .00009045093728 %1 x2 - .00001271966305 %1
+ .00006783820296 %1 x + .9421972632 10-6 , otherwise
%1 := Heaviside(x - .3750000000)
%2 := Heaviside(x - .2500000000)
STUDENT > Y(x):=piecewise(x>Lm,Yc(x),Ym(x));
Y(x) := { -.00001507515621 Heaviside(x) x3 + .00004522546863 %2 x3 - .7066479474 10-6 %2
+ .8479775370 10-5 %2 x - .00003391910148 %2 x2 - .00003015031243 %1 x3
+ .1589957880 10-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x2
+ .9421972632 10-6 x , .2500000000 < x
- .00001507515621 Heaviside(x) x3 + .00004522546863 %2 x3 - .7066479474 10-6 %2
+ .8479775370 10-5 %2 x - .00003391910148 %2 x2 - .00003015031243 %1 x3
+ .1589957880 10-5 %1 - .00001271966305 %1 x + .00003391910148 %1 x2
+ .9421972632 10-6 x , otherwise
%1 := Heaviside(x - .3750000000)
%2 := Heaviside(x - .2500000000)
STUDENT >
STUDENT > # Plot results
STUDENT > plot(T(x),x=0..Lm+Le,title='SlopeRadians');

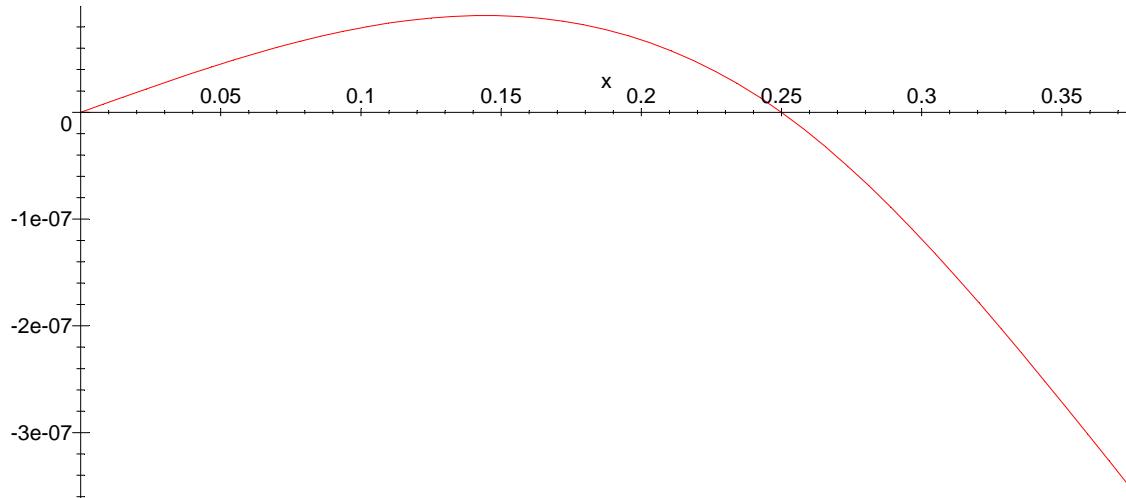
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### SlopeRadians



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STUDENT > plot(Y(x),x=0..Lm+Le,title='Displacement_Meter');
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Displacement\_Meter



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STUDENT > # Find the angle at motor inboard bearing (T at x=Lm)
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STUDENT > AngleRadians:=evalf(subs(x=Lm,Te(x)));
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$$\text{AngleRadians} := -1.88439453 \times 10^{-5}$$

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STUDENT > AngleDegrees:=AngleRadians*evalf(180/Pi);
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$$\text{AngleDegrees} := -0.0001079678535$$

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STUDENT > AngleMinutes:=AngleDegrees*60;
```

$$\text{AngleMinute}$$

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Maple V Release 4 - Student Edition

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[ STUDENT > # Moment is negligible at this low angle per NSK figure 4
  page 136
[ STUDENT > #
  http://www.nsk.com/services/basicknowledge/technicalreport
  /05distribution.html
[ STUDENT >
[ STUDENT > # *** Conclusion - bearing reaction moment is not
  significant for this case, and probably not for any
  electric motor since the shaft is stout. It may become a
  factor only for very long thin shafts under high load
[ STUDENT >
[ STUDENT > # Compute Bending stress
[ STUDENT > M_NM:=evalf(subs(x=Lm,M(x))); # moment at ib brg in
  Newtons
  M_NM := -22.20000000
[ STUDENT > Sbending_PA:=M_NM*(De/2)/I_e; # Bending stress in PA
  Sbending_PA := -.1809018745 107
[ STUDENT > Sbending_PSI:=Sbending_PA/6895; # convert PA to PSI
  Sbending_PSI := -262.3667505

```