

Code Details - Body Flanges

Component: Front Head Flng At TS

ASME Section VIII-1 2013 App. 2 Bolted Flange With Ring Type Gaskets

Flange type: Integral tapered hub - ASME fig.2-4(6)

Flange material: SA-350 K03011 Grd LF2 Cls 2 Forgings

Int. design pressure	PI = 0.128 Kg/mm2	Design temperature	T = 250 C
Ext. design pressure	PE = 0.01 Kg/mm2	B1 = B+g1 or B+go	B1 = -
Inside corr. allow	CAI = 3 mm	Outside corr. all.	CAO = 0 mm
Stress (operating)	SFO = 13.83 Kg/mm2	Stress (atmos.)	SFA = 14.06 Kg/mm2
Outside diameter	A = 881 mm	Inside spherical rad.	L = -
Inside diameter	B = 756 mm	Hub thickness	g1 = 12.75 mm
Bolt circle diameter	C = 839 mm	Hub tks. at attach.	go = 9 mm
Mean gasket diameter	G = 794.15 mm	Weld leg/hub length	h = 30 mm
Hub to bolt circle	R = 28.75 mm	Bolt circle to OD	E = 21 mm
Flange thickness	t = 58 mm		
Overlay thickness	OL = -		

Gasket material: Grooved iron or soft metal

Gasket outside dia.	ODG = 807 mm	Gasket width	N = 13 mm
Gasket thickness	tk = 3.17 mm	Gasket factor	m = 4.000
Gasket seating stress	y = 7.04 Kg/mm2	Gasket eff. width	b = 6.42 mm
Gasket unit stress	Sg = 4.65 Kg/mm2	factor f	f = 0 mm
Gasket rib length	Rib = 781 mm	Seating width	bo = 6.5 mm
Gasket rib eff width	Br = 4.76 mm	(Table 2-5.2 facing 1a/1b Col. II)	

Bolt material: SA-320 G40370 Grd L7A Bolt(<= 2 1/2)

Bolt diameter	db = 19.05 mm	No. of bolts	n = 44
Bolt root area	Area = 194.84 mm2	$Sg = Ab*Sa / ((\pi/4) * ((do-f)^2 - di^2))$	
$Bsmax = 2*db + 6*t / (m + 0.5)$		Actual bolt spacing	Bs = 59.9 mm
Max bolt spacing	BsMax = 115.4 mm	Min bolt spacing	BsMin = 44.5 mm
$Cf = \sqrt{Bs/Bsmax}$	Cf = 0.72	Cf used	Cf = 1
Stress (operating)	SB = 17.58 Kg/mm2	Stress (atmos.)	SA = 17.58 Kg/mm2

Bolting calculations:

Joint-contact compr. load	HP = 6.2832*b*G*PI*m+2*Br*m*PI*RIB	=	20222 kgf
Hydrostatic end force	H = 0.7854*G*G*PI	=	63403 kgf
Hydrostatic end force	H = 0.7854*G*G*PE	=	4953 kgf

Operating conditions:

Min. calc. bolt load	WM1 = HP+H	=	83625 kgf
Min. used bolt load	WM1 = max of 2 mating flanges	=	107141 kgf

Bolting up conditions:

Minimum bolt load	WM2 = 3.1416*b*G*Y+Br*Y*RIB	=	138949 kgf
Min. used bolt load	WM2 = max of 2 mating flanges	=	138949 kgf
Required bolt area	AM = WM2/SA or WM1/SB	=	7905.3 mm2
Available bolt area	AB = No.Bolt*Area	=	8572.9 mm2
Ratio of bolt areas	AB/AM = 1.084		
Design bolt load	W = 0.5*(AM+AB)*SA	=	144816 kgf

Minimum gasket width	NMIN = AB*SA / (6.283*y*G)	=	4.29 mm
Gasket compression stress	Gcst = AB*SA / ((\pi*G*N)+(Br*RIB))	=	4.17 Kg/mm2

Loads: Integral Flange Calculations

Operating conditions:

Hydrostatic end load	HD = 0.785*B*B*PI	=	57457 kgf
Hydrostatic end load	HDe= 0.785*B*B*PE	=	4489 kgf
Gasket load	HG = WM1-H	=	43739 kgf
Result. hydrostatic force	HT = H-HD	=	5945 kgf
Result. hydrostatic force	HTe= He-HDe	=	464 kgf

Bolting up conditions:

Gasket load	HG = W	=	144816 kgf
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Operating conditions:

Hydrostatic lever arm	hd = R+0.5*g1	=	35.13 mm
Gasket load lever arm	hg = (C-G)/2.	=	22.42 mm
Result. hydro. lever arm	ht = (R+g1+hg)/2.0	=	31.96 mm

Bolting up conditions:

Gasket load lever arm	hg = (C-G)/2.	=	22.42 mm
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Operating conditions:

Hydrostatic moment	MD = HD*hd	=	2018 kgf*m
Gasket moment	MG = HG*hg	=	981 kgf*m
Result. hydro. moment	MT = HT*ht	=	190 kgf*m
Total operating moment	MOP = MD+MG+MT	=	3189 kgf*m
	MOPe = HDe(hd-hg)+HTe(ht-hg)	=	61 kgf*m

Bolting up conditions:

Bolt up moment	MATM = W*hg	=	3247 kgf*m
Effective bolt moment	MB = MATM*SFO/SFA	=	3194 kgf*m

Total moment

MO = MOP or MB	=	3194 kgf*m
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Bolt spacing correction

M = MO*Cf	=	3194 kgf*m
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(TEMA 2007 RCB-11.23) Cf= 1

Flange shape constants:

K = A/B	= 1.1653	ho = SQ(B*G0)	= 82.4864
T = Fig.2-7.1	= 1.8527	h/ho = h/ho	= 0.3637
Z = Fig.2-7.1	= 6.5862	F = Fig.2-7.2	= 0.8732
Y = Fig.2-7.1	= 12.7582	V = Fig.2-7.3	= 0.3685
U = Fig.2-7.1	= 14.02	f = Fig.2-7.6	= 1.0
G1/G0 = G1/Go	= 1.4167	e = F/ho	= 0.0106
t =	= 58 mm		
d = U*ho*g0*g0/V	= 254220	Alpha = t*e+1.0	= 1.614
Beta = 1.333*t*e+1.0	= 1.8185	Gamma = Alpha/TF	= 0.8712
Delta = t*t*t/D	= 0.7675	Lambda = Gamma+Delta	= 1.6386

Stress calculations:

Allowable stress:

Long. hub	SH = (f*M)/(Lambda*g1**2*B)	=15.86 Kg/mm2	1.5*SFO =21.09 Kg/mm2
Radial	SR = Beta*M/(Lambda*t**2*B)	= 1.39 Kg/mm2	SFO =13.83 Kg/mm2
Tangential	ST1 = M*Y/(t**2*B) - (Z*SR)	= 6.84 Kg/mm2	SFO =13.83 Kg/mm2
(greater)	ST2 = (SH+SR)/2 or (SH+ST1)/2	=11.35 Kg/mm2	SFO =13.83 Kg/mm2

Component: Front Head Flng At TS

ASME Section VIII Div.1 2013, Appendix 2, 2-14 Flange Rigidity

--- Calculations ---

Operating moment,	Mo = 3189 kgf*m	Gasket seat. moment	Ma = 3247 kgf*m
Factor VI	VI = 0.3685	Factor L	L = 1.6386
Mod. elast.design T	Ed = 19129 Kg/mm2	Mod.elast.atm. temp	Ea = 20530 Kg/mm2
Thickness g0	g0 = 9 mm	Factor h0	h0 = 82.5 mm
Factor KI	KI = 0.3	Factor KL	KL = 0.2
Corrosion allowance	ca = 3 mm	Factor K	K = 1.1653
Thickness, T	T = 58 mm		

Flange Rigidity

Loose type flanges without hubs and optional flanges designed as loose type

Gasket seating $J = 109.4 * Ma / (E * T ** 3 * Ln(K) * KL) = -$

Operating $J = 109.4 * Mo / (E * T ** 3 * Ln(K) * KL) = -$

Integral type flanges and optional type flanges designed as integral and

Loose type flanges with hubs

Gasket seating $J = 52.14 * Ma * VI / (L * E * G0 ** 2 * ho * KI) = 0.9253$

Operating $J = 52.14 * Mo * VI / (L * E * G0 ** 2 * ho * KI) = 0.9751$

ASME appendix 2 calculation of hub thickness 'go' as a cylinder

Design pressure	P = 0.128 Kg/mm2	Allowable stress	S = 13.83 Kg/mm2
Outside radius	OR = -	Inside radius	IR = 378 mm
Joint efficiency	E = 0.85	Corr.Allow or OL	c = 3 mm
		Material tolerance tol	tol = 0 mm

Min hub thk / small end = $P * IR / (S * E - 0.6 * P) + c + Tol$ UG-27(c) (1)
= 7.14 mm

Hub thk / small end = 12 mm

New thickness 'go'	= 12 mm	New thickness 'g1'	= 15.75 mm
Corroded thickness 'go'	= 9 mm	Corroded thk 'g1'	= 12.75 mm

Code Details - MAWP/MDMT/Test P/Static P

Maximum Allowable Working Pressures

* = Shell Side MAWP + = Tube Side MAWP

Component	Side	--Design conditions--			---- New and cold ----		
		Temp C	Stress kg/mm2	MAWP kg/cm2	Temp C	Stress kg/mm2	MAWP kg/cm2
Shell Cylinder	S	220	14.06	28.06	21.1	14.06	37.53
Front Head Cylinder	T	250	14.06	28.06	21.1	14.06	37.53
Front Head Cover	T	250	14.06	33.4	21.1	14.06	44.5
Shell Cover	S	220	14.06	33.4	21.1	14.06	44.5
Front Tubesheet	S	250	13.83	18.35	21.1	14.06	22.72*
Front Tubesheet	T	250	13.83	18.35	Flange MAWP		
Front Head Flng At TS	T	250	13.83	16.62			
Front Shell Flng	S	220	13.98	16.97*	21.1	14.06	24
Tubes	T	250	9.42	225.13	21.1	9.42	225.13
Nozzle N1	T	250	12.02	13.95+	21.1	12.02	24.97
Nozzle N2	T	250	13.83	515.58	21.1	14.06	661.18
Nozzle N3A	S	220	12.02	22.02	21.1	12.02	33.09
Nozzle N3B	S	220	12.02	22.02	21.1	12.02	33.09
Nozzle N4A	S	220	12.02	22.02	21.1	12.02	33.09
Nozzle N4B	S	220	12.02	22.02	21.1	12.02	33.09
Nozzle Flng N1	T	250	13.83	42.92	21.1	14.06	52.03
Nozzle Flng N2	T	250	13.83	42.92	21.1	14.06	52.03
Nozzle Flng N3A	S	220	13.98	44.05	21.1	14.06	52.03
Nozzle Flng N3B	S	220	13.98	44.05	21.1	14.06	52.03
Nozzle Flng N4A	S	220	13.98	44.05	21.1	14.06	52.03
Nozzle Flng N4B	S	220	13.98	44.05	21.1	14.06	52.03
Nozzle Reinforcement N1	T	250	-	17.28	21.1	-	22.95
Nozzle Reinforcement N2	T	250	-	33.31	21.1	-	44.16
Nozzle Reinforcement N3A	S	220	-	18.63	21.1	-	24.65
Nozzle Reinforcement N3B	S	220	-	18.63	21.1	-	24.65
Nozzle Reinforcement N4A	S	220	-	18.63	21.1	-	24.65
Nozzle Reinforcement N4B	S	220	-	18.63	21.1	-	24.65
Front Hd Bolting At TS	T	250	17.58	23.06	Bolting for the same flange MAWP		
Front Hd Bolting At TS	S	250	17.58	23.35			
Nozzle Flng Bolting N1	T	250	17.58	42.92			
Nozzle Flng Bolting N2	T	250	17.58	42.92			
Nozzle Flng Bolting N3A	S	220	17.58	44.05	21.1	17.58	52.03
Nozzle Flng Bolting N3B	S	220	17.58	44.05	21.1	17.58	52.03
Nozzle Flng Bolting N4A	S	220	17.58	44.05	21.1	17.58	52.03
Nozzle Flng Bolting N4B	S	220	17.58	44.05	21.1	17.58	52.03