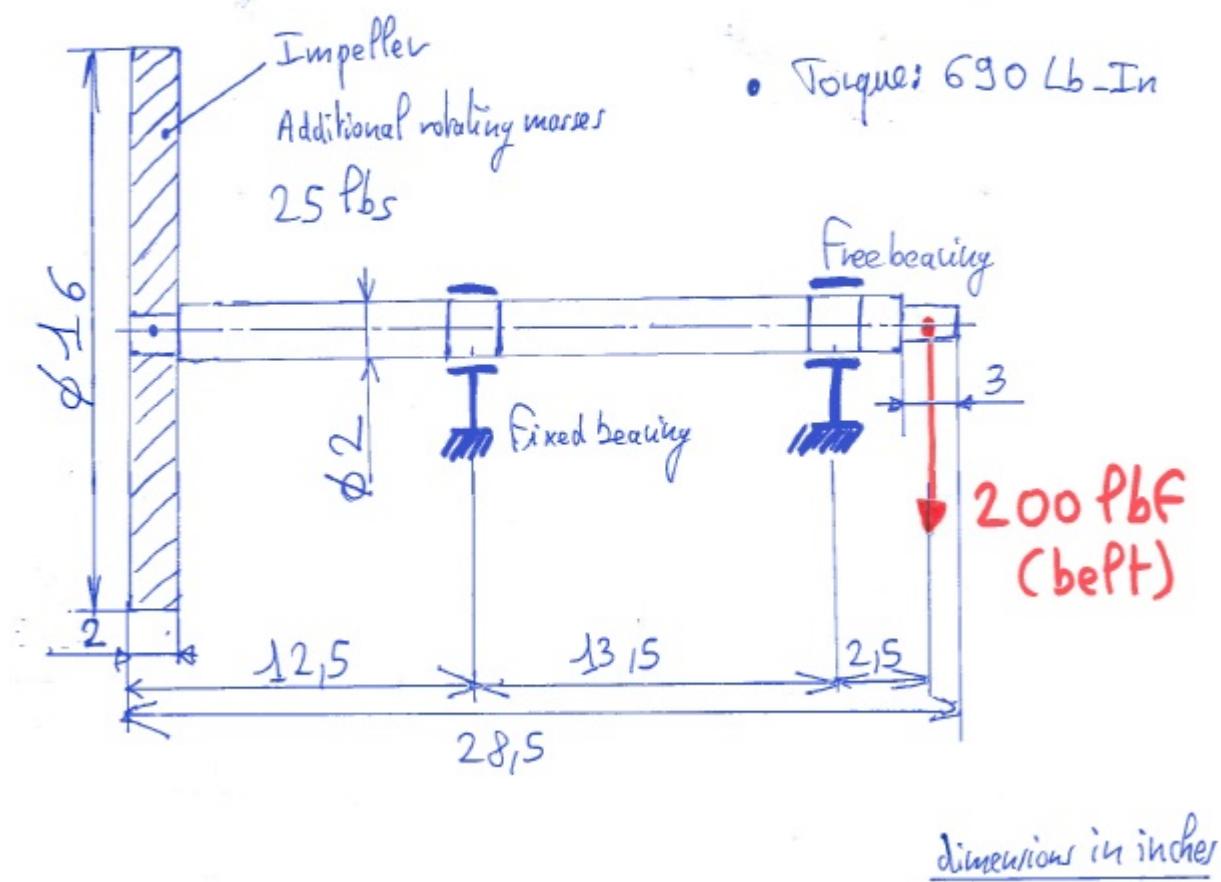
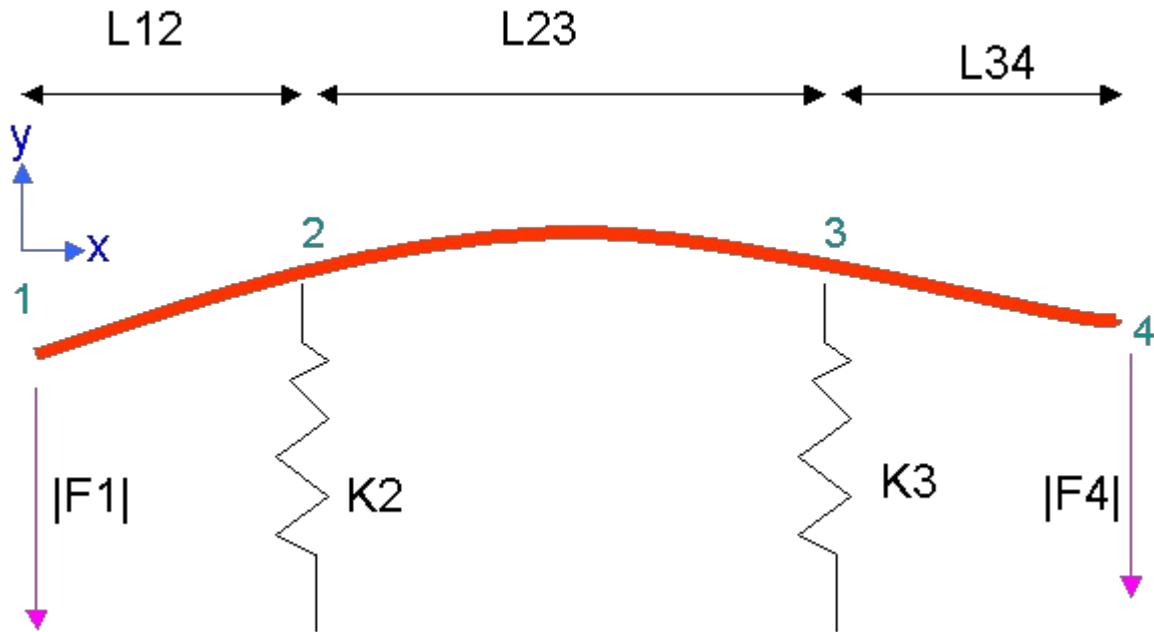


Model system to analyse Static Deflections:
Version 15: $K_2 = K_3 =$

• 1800 rpm



Simplified Model. Neglect weight of shaft



Note displacements and forces are positive in the upward direction. The applied forces shown happen to have negative values.

=====Symbols=====

x = longitudinal coordinate

y = radial coordinate

$V(x)$ = Shear due to force F_1

$M(x)$ = Moment due to force F_1

$T(x)$ = Slope (Θ) due to force F_1

$Y(x)$ = Displacement due to force F_1

$VV(x)$ = Shear due to force F_2

$MM(x)$ = Moment due to force F_2

$TT(x)$ = Slope (Θ) due to force F_2

$YY(x)$ = Displacement due to force F_2

Initialization:

```
[reset()
[assume(L12>0); assume(L23>0); assume(L34>0); assume(F1>0);
[assume(K2>0); assume(K3>0);
use(linalg)
Warning: 'htranspose' already has a value, not exported. [use]
Warning: 'transpose' already has a value, not exported. [use]
Warning: 'det' already has a value, not exported. [use]
```

==== PART 1 - Find deflections due to force F_1 =====

For the first part, we want to determine α_{11} and α_{14} .

We will apply force F_1 upwards at point 1 and compute the response displacements Y_1 and Y_4 . Then $\alpha_{11} = Y_1/F_1$ and $\alpha_{14} = Y_4/F_1$

Note we must assume $F_4=0$ during this part! (That's a helpful simplifier)

Find R_3 based on moment balance about point 2

$$R_3 := \text{solve}(F_1 * L_{12} = R_3 * L_{23}, R_3) [1]$$
$$\frac{F_1 L_{12}}{L_{23}}$$

Find R_2 based on Force balance:

$$R_2 := \text{solve}(F_1 + R_2 + R_3 = 0, R_2) [1]$$
$$- F_1 - \frac{F_1 L_{12}}{L_{23}}$$

Double check the moment balance about point 3 is satisfied

$$\text{simplify}(F_1 * (L_{12} + L_{23}) + R_2 * L_{23})$$
$$0$$

Double check passes!

Check forces sum to zero

$$F_1 + R_2 + R_3$$
$$0$$

Check passes!

Write the Shear function as integration of applied forces

$$V(x) := \text{simplify}(F_1 * \text{heaviside}(x) + R_2 * \text{heaviside}(x - L_{12}) + R_3 * \text{heaviside}(x - (L_{12} + L_{23}))$$
$$\frac{F_1 (L_{23} \text{heaviside}(x) - L_{23} + L_{12} \text{heaviside}(L_{12} - x) + L_{23} \text{heaviside}(L_{12} - x) - L_{12} \text{heaviside}(L_{12} + L_{23}))}{L_{23}}$$

Write Moment as integration of shear, with unknown integ constant "M0"

$$M(x) := \text{simplify}(\text{int}(V(x), x)) + M_0$$
$$M_0 - F_1 x - F_1 \text{heaviside}(L_{12} - x) (L_{12} - x) + F_1 x \text{heaviside}(x) - \frac{F_1 L_{12} \text{heaviside}(L_{12} - x) (L_{12} - x)}{L_{23}} + \dots$$

Solve M_0 based on B.C. that moment is zero at/beyond point 3

$$M_0 := \text{solve}((M(x) | x=L_{12} + L_{23}) = 0, M_0) [1]$$
$$0$$

As a double check, make sure the moment at left end is also zero:

$$M(x) | x=0$$
$$\frac{F_1 L_{12} (L_{12} + L_{23})}{L_{23}} - \frac{F_1 L_{12}^2}{L_{23}} - F_1 L_{12}$$

Check passes

[

Solve slope $T(x)$ (slope) by integrating $M(x)$ and multiplying $1/EI$, with integ const "T0"

$$T(x) := \text{int}(M(x), x) / EI + T0$$

$$T0 + \frac{\frac{F1 x^2 \text{heaviside}(x)}{2} - \frac{F1 x^2}{2} + \frac{F1 \text{heaviside}(L12-x) (L12-x)^2}{2} - \frac{F1 L12 \text{heaviside}(L12+L23-x) (L12+L23-x)^2}{2 L23} + \frac{F1 L12 \text{heaviside}(L12-x) (L12-x)^2}{2}}{EI}$$

$$T(x) := \text{simplify}(T(x))$$

$$T0 + \frac{\frac{F1 x^2 \text{heaviside}(x)}{2} - \frac{F1 x^2}{2} + \frac{F1 \text{heaviside}(L12-x) (L12-x)^2}{2} - \frac{F1 L12 \text{heaviside}(L12+L23-x) (L12+L23-x)^2}{2 L23} + \frac{F1 L12 \text{heaviside}(L12-x) (L12-x)^2}{2}}{EI}$$

$$\text{// } T(x) := \text{simplify}(1/EI/I1 * \text{int}(M(x), x)) + T0$$

Solve $Y(x)$ as integral of $T(x)$, with integration const "Y0"

$$Y(x) := \text{int}(T(x), x) + Y0$$

$$Y0 + T0 x - \frac{F1 x^3}{6 EI} + \frac{F1 x^3 \text{heaviside}(x)}{6 EI} - \frac{F1 \text{heaviside}(L12-x) (L12-x)^3}{6 EI} - \frac{F1 L12 \text{heaviside}(L12-x) (L12-x)^3}{6 EI L23}$$

$$Y(x) := \text{simplify}(Y(x))$$

$$Y0 + T0 x - \frac{F1 x^3}{6 EI} + \frac{F1 x^3 \text{heaviside}(x)}{6 EI} - \frac{F1 \text{heaviside}(L12-x) (L12-x)^3}{6 EI} - \frac{F1 L12 \text{heaviside}(L12-x) (L12-x)^3}{6 EI L23}$$

Solve Y_0, T_0 using force/displacement equations at points 2 and 3

$$\text{eq1} := (Y(x) | (x=L12)) = -R2/K2$$

$$Y0 + L12 T0 + \frac{F1 L12 L23^2}{6 EI} = \frac{F1 + \frac{F1 L12}{L23}}{K2}$$

$$\text{eq2} := (Y(x) | (x=L12+L23)) = - R3/K3$$

$$Y0 + T0 (L12 + L23) = - \frac{F1 L12}{K3 L23}$$

$$Y0 T0 \text{soln} := \text{simplify}(\text{solve}(\{\text{eq1}, \text{eq2}\}, \{Y0, T0\}));$$

$$\left\{ T0 = - \frac{F1 (-K2 K3 L12 L23^3 + 6 EI K3 L23 + 6 EI K2 L12 + 6 EI K3 L12)}{6 EI K2 K3 L23^2}, Y0 = \frac{F1 (6 EI K2 L12^2 + 6 EI K3 L12^2 + 6 EI K3 L23^2 - K2 K3 L12^2 L23^3 + 12 EI K3 L12 L23 - K2 K3 L12 L23)}{6 EI K2 K3 L23^2} \right.$$

$$Y0 := \text{simplify}(Y0 | Y0 T0 \text{soln})$$

$$\left. \frac{F1 (6 EI K2 L12^2 + 6 EI K3 L12^2 + 6 EI K3 L23^2 - K2 K3 L12^2 L23^3 + 12 EI K3 L12 L23 - K2 K3 L12 L23)}{6 EI K2 K3 L23^2} \right.$$

$$T0 := \text{simplify}(T0 | Y0 T0 \text{soln})$$

$$\left. - \frac{F1 (-K2 K3 L12 L23^3 + 6 EI K3 L23 + 6 EI K2 L12 + 6 EI K3 L12)}{6 EI K2 K3 L23^2} \right.$$

```

[ simplify(lhs(eq1)-rhs(eq1))
[ 0
[ simplify(lhs(eq2)-rhs(eq2))
[ 0

```

===== Part 2 - find equations for VV, MN, TT, YY from symmetry and similar approach as part 1

```

[ // Find R2 based on moment balance about point 3
[ R2:=solve(F4*L34=R2*L23,R2) [1]:
[ // Find R2 based on Force balance
[ R3:=solve(F4+R2+R3=0,R3) [1]:
[ VV(x):=simplify(F4*heaviside(x-(L12+L23+L34))+R2*heaviside(x-L12)+R3*heaviside(x-L23)):
[ MM(x):=simplify(int(VV(x),x))+MM0:
[ MM0:=solve((MM(x)|x=L12)=0,MM0) [1]:
[ TT(x):=int(MM(x),x)/EI+TT0:
[ TT(x):=simplify(TT(x)):
[ YY(x):=int(TT(x),x)+YY0:
[ YY(x):=simplify(YY(x)):
[ eq1:=(YY(x)|(x=L12))=-R2/K2:
[ eq2:=(YY(x)|(x=L12+L23))=-R3/K3:
[ YY0TT0soln:=simplify(solve({eq1,eq2},{YY0,TT0})):
[ YY0:=simplify(YY0|YY0TT0soln):
[ TT0:=simplify(TT0|YY0TT0soln):

```

Assign values in MKS system

```

[ L12:=11.5/39.37
[ 0.2921005842
[ L23:=13.5/39.37
[ 0.3429006858
[ L34:=2.5/39.37
[ 0.063500127
[ Ltot:=L12+L23+L34
[ 0.698501397
[ D1:=2/39.37 // diameter
[ 0.0508001016
[ F1:=-25*4.44

```

```

[ -111.0
F4:=-200*4.44
[ -888.0
E1:=2.3E11
[ 2.3 1011
I1:=float(PI)*D1^4/64
[ 0.0000003269100125
EI:=E1*I1
[ 75189.30287
K2:=1E10
[ 10000000000.0
K3:=K2
[ 10000000000.0
=====Plot=====
plot(plot::Function2d(V(x),x=0..Ltot,Color=RGB::Red,Legend = "Shear from F1",
plot::Function2d(VV(x),x=0..Ltot,Color=RGB::Blue,Legend = "Shear from F4"),
plot::Function2d(V(x)+VV(x),x=0..Ltot,Color=RGB::Black,LineStyle = Dashed,
,Header="Shear")

```

Shear

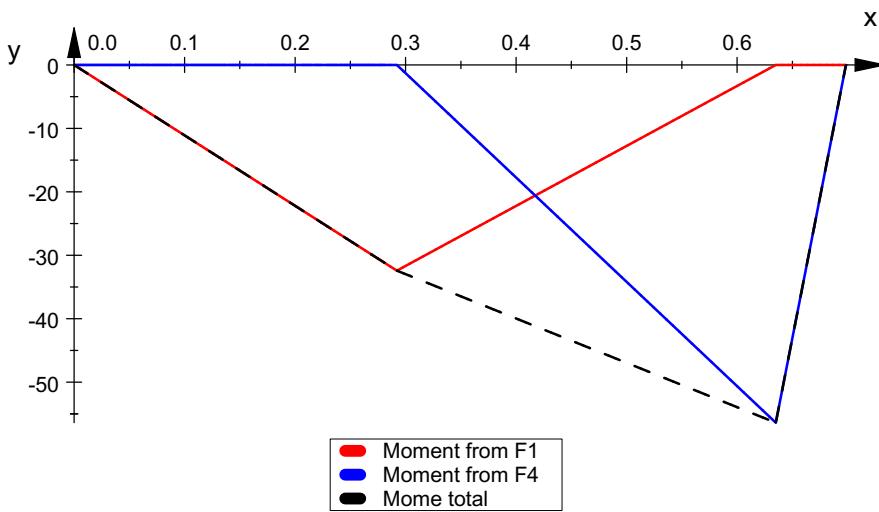
Shear from F1
Shear from F4
Shear total

```

plot(plot::Function2d(M(x),x=0..Ltot,Color=RGB::Red,Legend = "Moment from I",
plot::Function2d(MM(x),x=0..Ltot,Color=RGB::Blue,Legend = "Moment from F4"),
plot::Function2d(M(x)+MM(x),x=0..Ltot,Color=RGB::Black,LineStyle = Dashed,
,Header="Moment")

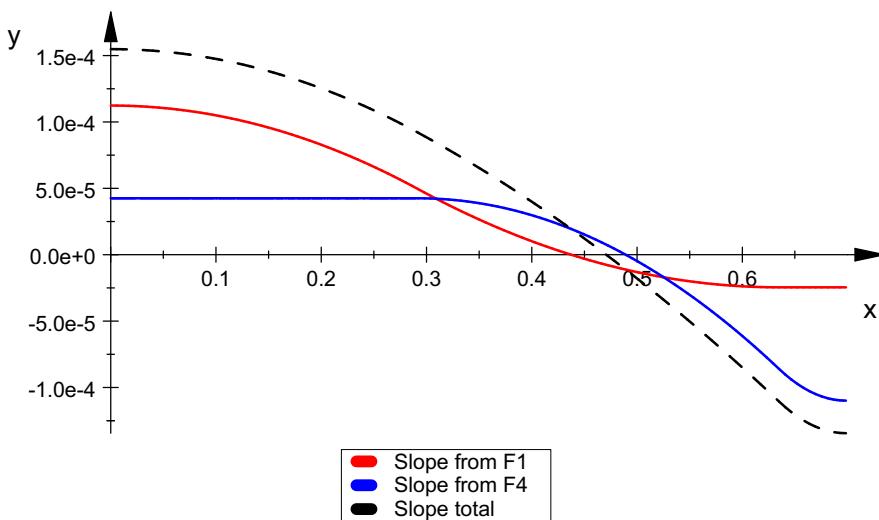
```

Moment



```
[ T(x):=simplify(T(x)|K23=K23): // needed simplification for plot to work
[ TT(x):=simplify(TT(x)|K23=K23): // needed simplification for plot to work
[ plot(plot::Function2d(T(x),x=0..Ltot,Color=RGB::Red,Legend = "Slope from F1",
plot::Function2d(TT(x),x=0..Ltot,Color=RGB::Blue,Legend = "Slope from F4"),
plot::Function2d(T(x)+TT(x),x=0..Ltot,Color=RGB::Black,LineStyle = Dashed,
I,Header="Slope")
```

Slope



```
[ Y(x):=simplify(Y(x))|L23=L23: // required for plot to work
[ YY(x):=simplify(YY(x))|L23=L23: // required for plot to work
[ plot(plot::Function2d(Y(x),x=0..Ltот,Color=RGB::Red,Legend = "Disp from F1",
plot::Function2d(YY(x),x=0..Ltот,Color=RGB::Blue,Legend = "Disp from F4"),
plot::Function2d(Y(x)+YY(x),x=0..Ltот,Color=RGB::Black,LineStyle = Dashed,
I,Header="Displacement, (K = ".expr2text(K2)." per brg)")
```

