

DATE : 21/01/16

PROJECT : 4 SLING STEEL LIFTING FRAME

Calculations done to BS2573-1-1983

Design Characteristics

Steel Strength (Y_s)	265 N/mm ²
Container Load (CL)	222.411 kN
Steel Section	305UKC118
Length of C_y	6100 mm
Length of C_x	2500 mm
Radius of Gyration of Section (R_y)	77.7 mm
Youngs Modulus E	205 Gpa
Area of section	15000 mm ²
Thickness of compression flange (T)	18.7 mm
Depth of section (D)	314.5 mm
Elastic Modulus X-X (z)	1760000 mm ³

Outcome

Analysis and Design

Determining Tension in Cables

Cables will have an angle of 45 degrees to the crane hook from the quadrants

$$T = CL/\sin 45$$

$$T = 314.5366526 \text{ kN}$$

$$\text{Axial Compression in } C_x : CL \cdot \cos(67.71)$$

$$C_x = 84.35930778 \text{ kN}$$

$$\text{Axial Compression in } C_y : CL \cdot \sin(67.71)$$

$$C_y = 205.7915453 \text{ kN}$$

Trying Steel Section : 305UKC118

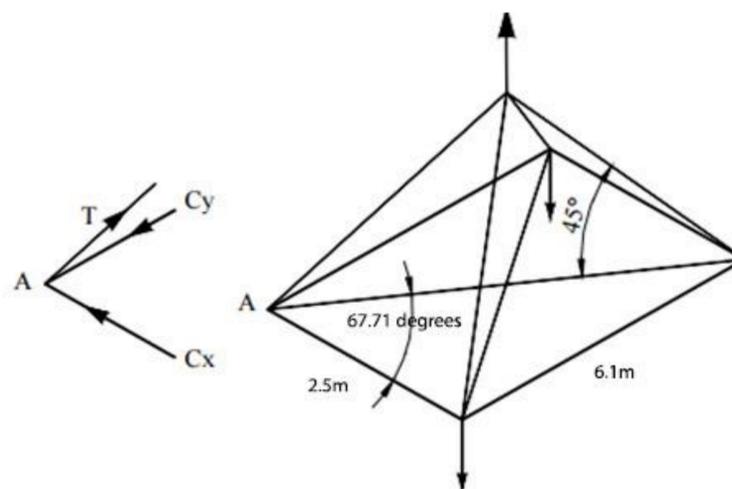


Figure 23 – Loading of Box Frame and Free body diagram

Calculating Compression Stress

According to BSI Members subjected to a combination of coexistent bending and axial compression, shall be so proportioned that:

$$(f_{ac}/P_{ac}) + (f_{bc}/P_{bc}) \leq 1$$

Where

f_{ac} is the calculated axial compressive stress

P_{ac} is the permissible compressive stress in axially loaded compression members

f_{bc} is the calculated maximum compressive stress in bending, using the lesser value when bending occurs about both axis

Axial compressive stress:

The axial compressive stress shall not exceed 60% of the yield strength or the value obtained from :

$$P_{ac} = 0.6F_{crip}$$

Where F_{crip} is the applied stress at failure of a member subjected to overall flexural buckling due to axial compress as given by:

Where

C_o is the Euler critical stress = $(\pi^2 * E) / s^2$

Y_s is the yield stress of the steel

E is the Youngs modulus

n is the Perry coefficient

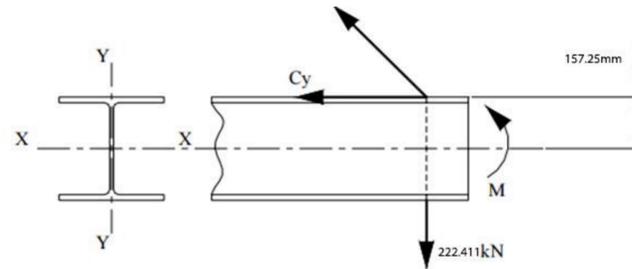
α is the Robertson constant from BSI Table 11 (taking as 5.5)

s is the slenderness ratio

s_o is the limiting slenderness ratio for stub columns taken as 17.5

r is radius of gyration

l_{eff} is the effective length relative to the same axis



$$l_{eff} = 0.7 * \text{Length of } C_y \\ = 4270 \text{ mm}$$

$$s = l_{eff} / r \\ = 54.95495495$$

$$n = \alpha (s - s_o) \times 10^{-3} \\ = 0.206002252$$

$$C_o = \pi^2 E / s^2 \\ = 669.9461474$$

$$F_{crip} = 204.40 \leq Y_s$$

Therefore:

$$P_{ac} = 0.6 * F_{crip} \\ = 122.64$$

$$f_{ac} = 13.72 \text{ N/mm}^2$$

Calculating Bending Stress

The permissible compressive stress due to bending must not exceed 62% of the yield stress or the value of P_{bc} corresponding to C_s

Section 5.1.4.2.3

The basic tensile bending stress P_{bt} shall not exceed $0.62Y_s$

Section 5.1.4.1

Using formula for I beam sections outlined in section 5.1.4.2.3 of BSI 2573

Where

C_s is the critical stress in the compression element

l is the effective length of the compression flange

r_y is the radius of gyration about the y-y axis at the point of maximum bending moment

D is the overall depth of the member at the point of maximum bending moment.

T is the effective thickness of the compression flange

$$C_s = \frac{1073.92}{1330.04} = 0.67715693$$

$$\begin{aligned} \text{Therefore } P_{bc} &= 0.62 \cdot Y_s \\ &= 164.30 \text{ N/mm}^2 \end{aligned}$$

calculated bending stress $f_{bc} = M/z$

$$M = 32360720.50 \text{ Nmm}$$

$z =$ elastic modulus x-x

$$f_{bc} = 18.39 \text{ N/mm}^2$$

Hence

$$\begin{aligned} (f_{ac}/P_{ac}) + (f_{bc}/P_{bc}) &\leq 1 \\ &= 0.22 \leq 1 \end{aligned}$$

PASS