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NOTE

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> ANALYZED PREVENTING EXPOSED WATER PIPES FROM FREEZING

> > by

D.G. Stephenson

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PREVENTING EXPOSED WATER PIPES FROM **FREEZING**

by

D.G. **Stephenson**

A water pipe that is exposed to an environment at temperatures lower than the freezing **point of** water **will not necessarily freeze eyen without insulation if there is a continuaus flow through** it, **but when there is no flow it will freeze regardless of insulation. The Lequired** minimum **flow rate depend8 on the temperature of the water entering the exposed section of pipe and the resistance to heat transfer** from **the water to** the environment. This note presents an equation relating these parameters.
It can be used to solve for any one of the three variables when the other **two are knawn. Idormation is also presented which facilitates the calculation of the thermal resistance between water in the pipe** and **the environment, for conditions that could cause freezing.**

Basic Eauation for Heat Loss

The rate of heat transfer from fluid **flowing through a pipe is:**

$$
Q = \frac{L \cdot \Theta_{\text{mean}}}{R_{\text{Total}}}
$$

where R_{Total} is the resistance to heat flow per unit length of pipe,

- **L is the length of the exposed** eection **of pipe,**
- **0 is** the **difference between the fluid temperature and the ambient air temperature.**

Let T_{in} = temperature of fluid entering pipe,

 T_{out} = temperature of fluid leaving pipe,

^T= **temperature of the environment. ambient**

Then

 θ_{in} = T_{in} - T_{ambient} θ_{out} = T_{out} - T_{ambient} $\theta_{\text{mean}} = \frac{\theta_{\text{in}} - \theta_{\text{out}}}{\theta_{\text{in}} / \theta_{\text{out}}} = \frac{T_{\text{in}} - T_{\text{out}}}{\theta_{\text{in}} / \theta_{\text{out}}}$ m **i n** $\frac{1}{2}$ **l n** $\left(\theta_{\text{in}}/\theta_{\text{out}}\right)$ **i 1 n** $\left(\theta_{\text{in}}/\theta_{\text{out}}\right)$

The **heat loss can also be related to** the **fluid flow rate and the temperature drop between** inlet **and outlet.**

$$
Q = W C (T_{in} - T_{out})
$$

where W is mass flow rate through the pipe, C is the **specific heat of the fluid** (- **1.0 for watek).**

Combining the two independent expressions for Q gives:

$$
ln (\Theta_{in} / \Theta_{out}) = \frac{L}{WCR_{Total}}
$$

or $\Theta_{in} = \Theta_{out}$ e

Thermal Resistances

The **total resistance to heat flow** between **fluid flawing thraugh a pipe and the outside environment is the sum of the following four components** :

(a) Rinside' which depends on the rate of flow **and the inside diameter of** the pipe. Figure 1 gives the relationship for cold water flowing

through a long pipe.

 $R_{pipe} = \frac{\ln (O.D./ I.D.)_{pipe}}{2 \pi K_{pipe}}$ (b)

> **where 0.** D. **and** I. D. are **the outside and inside diameters of the pipe respectively and K is the** thermal **conductivity of the pipe material. pipe**

 $-2-$

(c) R_{insulation} =
$$
\frac{\ln (O.D./I.D.)_{insulation}}{2 \pi K_{insulation}}
$$

For **most pipe insulations the conductivity has a value of about 0.025** ~tu/hr **ft OF.**

(d) R_{outside} =
$$
\frac{1}{H_c + H_R}
$$

where H_c and H_R are the conductances per lineal foot of pipe due to **convection and radiation respectively.** H_c depends on the outside **diameter of the cylinder and the velocity of the air blowing across it.** The relationship is shown graphically in Figure 2. H_p depends on **the** diameter **of the** cylinder **and the emissivity of** the outer **surface.** For surfaces with a high value of emissivity, H_R is approximately **equal to twice** the **outside** diameter **expressed in feet.** This **is isually** small compared to H_c and a lower emissivity surface makes **it smaller still.**

Minimum **Water** Temperature

If **a pipe is to remain completely free of ice, its inside surface temperature must** not fall **below 32 'F, which means that the water** temperature must be above 32° F. The minimum value of T_{out} is given by:

$$
T_{out} = 32 + \left(\frac{R_{inside}}{R_{pipe} + R_{insulation} + R_{outside}}\right) \left(32 - T_{ambient}\right)
$$

or
$$
\theta_{\text{out}} = \left(\frac{R_{\text{Total}}}{R_{\text{pipe}} + R_{\text{insulation}} + R_{\text{outside}}}\right) \left(32 - T_{\text{ambient}}\right)
$$

Example Problem

Find the **required inlet water temperature to prevent freezing in a 500-ft length of 6-in. schedule 40 steel pipe covered by a l -in. layer of insulation.** The **minimum water flow rate will be 3 gallons/rnin. (1 800** lb/hr) **and the ambient conditions are -10°F with a 30-mile-per-hour wind.** Find the required inlet w
in a 500-ft length of 6-in. schedule 4
of insulation. The minimum water f
lb/hr) and the ambient conditions ar
Data: 6-in. schedule 40 steel pipe

Solution:

A. Calculation of Thermal Resistances

$$
\frac{W}{\pi \times 1. D.} = \frac{1800}{\pi \times 6.065/12} = 1134
$$

From Figure 1, R_{inside} = 0.25
R_{pipe} = 1n (6.625/6.065) / (2 $\pi \times 28$) = 0.0005 i.e., negligible
R_{insulation} = 1n (8.625/6.625) / (2 $\pi \times 0.025$) = 1.68
R_{outside} = $\frac{1}{H_c + H_R}$
Wind Speed x O.D. _{insulation} = 30 x $\frac{8.625}{12}$ = 21.6
.. From Figure 2, H_c = 19.8
H_R = $\frac{2 \times 8.625}{12}$ = 1.4
R_{outside} = $\frac{1}{19.8 + 1.4}$ = 0.05
R_{Total} = 0.25 + 1.68 + 0.05 = 1.98

B. Calculation of Water Outlet Temperature

$$
\theta_{\text{out}} = \left(\frac{R_{\text{Total}}}{R_{\text{pipe}} + R_{\text{insulation}} + R_{\text{outside}}}\right) \quad \left(32 - T_{\text{ambient}}\right)
$$

$$
= \frac{1.98}{1.73} \times 42 = 48 \text{ degrees}
$$

$$
T_{\text{out}} = T_{\text{ambient}} + \Theta_{\text{out}} = -10 + 48 = \frac{38 \text{°F}}{1.73}
$$

C. Calculation of Water Inlet Temperature

$$
\theta_{in} = \theta_{out} e
$$

\n
$$
L/WCR_{Total} = \frac{500}{1800 \times 1 \times 1.98} = 0.140
$$

\n
$$
\theta_{in} = 48 e^{0.140} = 55 \text{ degrees}
$$

\n
$$
T_{in} = T_{ambient} + \theta_{in} = -10 + 55 = \frac{45 \text{ F}}{1000 \times 100} = 0.140
$$

The problem might have been to determine the minimum allowable flow rate for a given inlet temperature, say $T_{in} = 48 °F$.

Solution:

- A. as in first example assuming that flow will be laminar and $R_{inside} = .25$ regardless of flow rate
- B. as in first example
- C. Calculation of Minimum Flow Rate

$$
L/WCR_{Total} = \ln (\Theta_{in} / \Theta_{out})
$$

\n
$$
\Theta_{in} = T_{in} - T_{ambient} = 48 + 10 = 58
$$

\n
$$
\ln (\Theta_{in} / \Theta_{out}) = \ln (58/48) = 0.190
$$

\n
$$
W = \frac{L}{CR_{Total} \ln(\Theta_{in} / \Theta_{out})} = \frac{500}{1 \times 1.98 \times 0.190} = \frac{1330}{100} \text{ lb/hr.}
$$

FIGURE 1

THERMAL RESISTANCE BETWEEN PIPE AND WATER FLOWING THROUGH IT

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