

ad, efficiency, and inability to meet design objectives.

been expended in recent years to provide a design code base for the design of pump intakes. This effort is reflected in the publication of ANSI/HI 9.8-1998, "Pump Intake Design." The following are the reasons for the requirement for additional material on pump intakes.

Design Standards

Pump intakes relied more on standards developed by the British Hydrographic Association [1] and early standards such as those published in various publications, however, they were, with discussions on them, essentially to the weak point of view. That should be minimized, that should be followed, or that should be consulted.

The Institute appointed a committee of designers and for the first time engineering consultants, and to improve the standards the first time, to consider them in detail. A new and improved standard was adopted and, after consensus, adopted as the American National Standard for Pump Intake Design.

The American National Standard for Pump Intake Design, ANSI/HI 9.8-1998, should have and does plus all of the other standards. They are available in ANSI standards. They are available in www.pumps.org, topic 9.8.

ANSI/HI 9.8 establishes a benchmark for engineers. It establishes a level of performance for all groups. Engineering organizations at large have had the experience from their groups to so they reflect (1) the experience and (2) the experience in the subject. The organizations such as NFPA are designed to protect the public and parties can be heard in the process (the standards)

represents the best available information at hand, devoid of bias toward any particular product or point of view. Individual committees are required to document their deliberations and to invite public comment on draft standards as a part of the process. Responses to public comment are documented and made a part of the record. The results, although not necessarily representing the cutting edge of available technology, do represent the minimum in acceptable performance, and that minimum should be considered when any project is developed. The owner can certainly decide whether the standard should be used, but the engineer has an obligation to inform the owner of the perils faced by nonconformance, including the prospect that (1) the installation may not perform as desired, and (2) warranties may be voided.

This book is complementary to and compatible with these standards, and the presentation in this chapter not only follows ANSI/HI 9.8-1998 but also reflects: (1) experience gained through more than 45 years of designing pumping station wet wells, (2) information developed at the U.S. Corps of Engineers Waterways Experiment Station in Vicksburg, Mississippi, (3) investigations at Montana State University at Bozeman, ENSR Hydraulic Laboratory in Redmond, Washington, Alden Research Laboratory in Holden, Massachusetts, and northwest hydraulic consultants in Seattle, Washington, and (4) full-scale tests at the Fairbanks Morse Pump Corp. plant in Kansas City—all of which have led to several publications [3, 4, 5, 6, 7].

Pump Inlet Conditions

Undesirable features noted in many sump designs include:

1. A free fall (no matter how short) from the inlet conduit into the sump or pool below with the consequent entrainment of air in the liquid and (with wastewater) the release of odors. The air bubbles, easily captured by currents and carried into the pumps, cause loss of capacity and damage to the equipment. Air discharged into pipes promotes sulfuric acid production, and in unprotected concrete pipes the combination leads to collapsing sewers.
2. Piping with excessive velocities that cause unreasonable headloss and can lead to vibration problems due to turbulence in fittings and valves.
3. Abrupt changes in flow direction upstream from the pump inlet connection. In sumps, abrupt changes usually cause vortices. In intake

manifolds and pump inlet piping, abrupt changes in direction may cause flow to become asymmetrical and thus overload pump shafts and bearings. Abrupt changes in flow direction are acceptable when the pump manufacturer (1) supplies the fitting as a part of the equipment, or (2) does not take exception to the presence of the fitting. Pump operation is probably not adversely affected in such circumstances.

4. Sump or inlet piping geometry that permits differential velocities and, thus, rotation of the fluid. With the slightest rotation, the spin increases as the water approaches the pump suction inlet. Swirling in the suction pipe may reduce the local NPSHA in the core to zero and thereby cause cavitation, noise, and rapid wear even though the average NPSHA is adequate. Swirling at the impeller changes the angle of attack of the flow to the impeller blades and shifts the pump curve, often drastically.
5. Horizontal velocities in sumps near the pump inlets that are too high. In general, such velocities should be less than 0.3 m/s (1 ft/s). Actual velocities usually differ greatly from calculated average velocities.
6. Interference between adjacent intakes. Space between intakes no closer than $2.5 D$ c-c (where D is intake bell diameter). Also consider access clearance [1.1 m (42 in.) minimum] between adjacent machines.
7. Discontinuities such as corners without fillets and uneven distribution of currents caused by flow past pier noses that often result in the formation of air-entraining vortices. Although there is usually no surface indication, subsurface vortices may also occur, and they can be very damaging.
8. Stagnant areas in wastewater pumping station wet wells where velocities are too low to prevent the deposition of the putrescible solids in wastewater. Velocities of about 0.3 m/s (1 ft/s) are enough to keep organic solids moving, whereas velocities in excess of 0.5 m/s (2 ft/s) are required to keep grit moving. After organic material and grit are deposited, velocities in excess of 1.6 m/s (5 ft/s) are required to move them with reasonable celerity.

See Section 27-6 for a discussion of wet well failures in which the designer did not follow the above advice.

The water velocity into pump intakes recommended in ANSI/HI 9.8-1998 is given in Table 12-1. However, the authors recommend the velocity be limited to about 1.1–1.2 m/s (3.5–4.0 ft/s). This rec-

