$$
R_a = \frac{1}{2} \cdot \omega \cdot L \qquad \dots \text{ support reaction load (vertical force equilibrium)}
$$
\n
$$
M(x) = R_a \cdot x - \frac{1}{2} \cdot \omega \cdot x^2 \qquad \dots \text{ BM along beam length, as a function of x}
$$
\n
$$
M(x) = \frac{1}{2} \cdot \omega \cdot L^2 \cdot \left[ \frac{x}{L} - \left( \frac{x}{L} \right)^2 \right] \qquad \dots \text{ BM, substituting for } R_a
$$
\n
$$
y_{xx}(x) = \frac{M(x)}{E \cdot L}
$$
\n
$$
y_x(x) = y_{xA} + \frac{\omega \cdot L^2}{2 \cdot E \cdot L} \cdot \int_0^{x} \frac{x}{L} - \left( \frac{x}{L} \right)^2 dx \qquad \dots \text{ gradient integral}
$$
\n
$$
y_x(x) = y_{xA} + \frac{\omega \cdot L^3}{12 \cdot E \cdot L} \cdot \left[ 3 \cdot \left( \frac{x}{L} \right)^2 - 2 \cdot \left( \frac{x}{L} \right)^3 \right] \qquad \dots \text{ gradient as function of x}
$$
\n
$$
y_x\left(\frac{L}{2}\right) = 0 = y_{xA} + \frac{\omega \cdot L^3}{24 \cdot E \cdot L} \qquad \dots \text{ set gradient to zero at } x = L/2
$$
\n
$$
y_{xA} = \frac{\omega \cdot L^3}{24 \cdot E \cdot L} \qquad \dots \text{ same as Roark } \text{ Th Edition.}
$$
\n
$$
y_x(x) = -\frac{\omega \cdot L^3}{24 \cdot E \cdot L} \quad \dots \text{ same as Roark } \text{ Th Edition.}
$$
\n
$$
y_x(x) = -\frac{\omega \cdot L^3}{24 \cdot E \cdot L} \cdot \int_0^{x} 1 - 6 \cdot \left( \frac{x}{L} \right)^2 + 4 \cdot \left( \frac{x}{L} \right)^3 \qquad \dots \text{ gradient as function of x}
$$
\n
$$
y(x_p) = -\frac{\omega \cdot L^3}{24 \cdot E \cdot L} \cdot \int_0^{x_p} 1 - 6 \cdot \left( \frac{x}{L} \right)^2 + 4 \cdot \left( \frac{x}{L} \right)^3 dx \qquad \dots \text{ displacement integral}
$$

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$$
y(x) = -\frac{\omega L^3 x}{24 \cdot E \cdot 1} \left[ 1 - 2 \left( \frac{x}{L} \right)^2 + \left( \frac{x}{L} \right)^3 \right] \dots \text{displacement as function of x}
$$
\n
$$
y_{\text{max}} = y \left( \frac{L}{2} \right) = -\frac{5 \cdot \omega L^3}{384 \cdot E \cdot 1} \dots \text{ same as Roark } 7 \text{th Edition},
$$
\n
$$
\delta L = \frac{1}{2} \cdot \int_0^L \left( \frac{d}{dx} y(x) \right)^2 dx \dots \text{ change in beam length}
$$
\n
$$
\text{where } \dots \qquad \frac{d}{dx} y(x) = -\frac{\omega L^3}{24 \cdot E \cdot 1} \left[ 1 - 6 \cdot \left( \frac{x}{L} \right)^2 + 4 \cdot \left( \frac{x}{L} \right)^3 \right]
$$
\n
$$
\delta L = \frac{1}{2} \cdot \left( \frac{\omega L^3}{24 \cdot E \cdot 1} \right)^2 \cdot \int_0^L \left[ 1 - 6 \cdot \left( \frac{x}{L} \right)^2 + 4 \cdot \left( \frac{x}{L} \right)^3 \right]^2 dx
$$
\n
$$
\delta L = \frac{1}{2} \cdot \left( \frac{\omega L^3}{24 \cdot E \cdot 1} \right)^2 \cdot \left( \frac{17}{35} \cdot L \right)
$$
\n
$$
L := 160 \cdot \text{in} \qquad \omega := 2000 \cdot \frac{\text{lbf}}{L} \qquad \omega = 12.5 \cdot \frac{\text{lbf}}{\text{in}}
$$
\n
$$
E := 29 \cdot \text{Mpsi} \qquad I := \frac{1}{12} \cdot (2 \cdot \text{in})^4
$$
\n
$$
\delta L := \frac{1}{2} \cdot \left( \frac{\omega L^3}{24 \cdot E \cdot 1} \right)^2 \cdot \left( \frac{17}{35} \cdot L \right) \qquad \delta L = 0.118 \text{ in}
$$

 $\mathsf{l}$ 

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