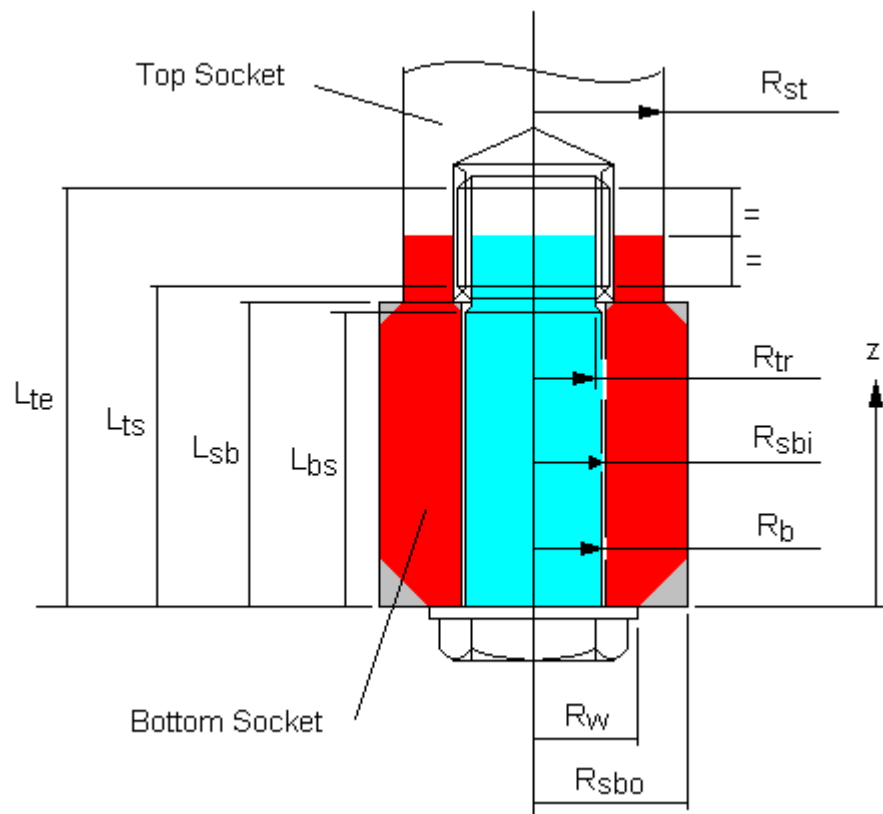


Preloaded Socket and Bolt Joint Under Applied External Loading



Input Data

Moduli ...	$E_b := 210 \cdot \text{GPa}$	... for bolt
	$E_{sb} := 200 \cdot \text{GPa}$	... for bottom socket
	$E_{st} := 190 \cdot \text{GPa}$	... for top socket
Lengths ...	$L_{sb} := 100 \cdot \text{mm}$	... bottom socket length
	$L_{bs} := 97 \cdot \text{mm}$	... bolt shank length
	$L_{ts} := 103 \cdot \text{mm}$	... length to start of thread
	$L_{te} := 180 \cdot \text{mm}$	... length to end of thread

Radii ...	$R_b := 25 \cdot \text{mm}$	... bolt shank radius
	$R_{tr} := 22.5 \cdot \text{mm}$	... bolt undercut radius
	$R_{sbo} := 53 \cdot \text{mm}$	... bottom socket outer radius
	$R_{sbi} := 25.5 \cdot \text{mm}$	... bottom socket inner radius
	$R_{st} := 45 \cdot \text{mm}$	... top socket outer radius
	$R_w := 40 \cdot \text{mm}$	... washer radius
	$R_{br} := 22.9 \cdot \text{mm}$	... bolt male thread root radius, note $R_{br} > R_{tr}$
	$R_{str} := 24.5 \cdot \text{mm}$	... top socket female thread root radius

Moment ... $M_{apl} := 11.50 \cdot \text{kN} \cdot \text{m}$... applied BM at interface between top and bottom sockets

Load ... $P_{apl} := 0 \cdot \text{kN}$... applied axial load

$P_{pl} := 600 \cdot \text{kN}$... joint preload

Socket Inner Radius Profiling

$$z_{si1} := L_{sb} - (\max(R_{str}, R_{sbi}) - \min(R_{str}, R_{sbi})) \quad z_{si1} = 99.00 \text{ mm}$$

$$z_{si2} := 0.5 \cdot (L_{ts} + L_{te}) \quad z_{si2} = 141.50 \text{ mm}$$

$$\text{Skt_In} := \begin{pmatrix} 0 \cdot \text{mm} & R_{sbi} \\ z_{si1} & R_{sbi} \\ L_{sb} & R_{str} \\ z_{si2} & R_{str} \end{pmatrix} \quad \text{Skt_In} = \begin{pmatrix} z & r \text{ (mm)} \\ 0 & 25.5 \\ 99 & 25.5 \\ 100 & 24.5 \\ 141.5 & 24.5 \end{pmatrix} \text{ mm} \quad \dots \text{ interpolating socket inner radius}$$

$$r_{si}(z) := \text{linterp}(\text{Skt_In}^{\langle 0 \rangle}, \text{Skt_In}^{\langle 1 \rangle}, z) \quad \dots \text{ socket inner radius function}$$

Socket Outer Radius Profiling

$$z_{so1} := R_{sbo} - R_w \quad z_{so1} = 13.00 \text{ mm}$$

$$z_{so2} := L_{sb} - (R_{sbo} - R_{st}) \quad z_{so2} = 92.00 \text{ mm}$$

$$z_{so3} := 0.5 \cdot (L_{ts} + L_{te}) \quad z_{so3} = 141.50 \text{ mm}$$

$$\text{Skt_Out} := \begin{pmatrix} 0 \cdot \text{mm} & R_w \\ z_{so1} & R_{sbo} \\ z_{so2} & R_{sbo} \\ L_{sb} & R_{st} \\ z_{so3} & R_{st} \end{pmatrix} \quad \text{Skt_Out} = \begin{matrix} z & r \text{ (mm)} \\ \begin{pmatrix} 0 & 40 \\ 13 & 53 \\ 92 & 53 \\ 100 & 45 \\ 141.5 & 45 \end{pmatrix} \end{matrix} \text{ mm} \quad \dots \text{ interpolating socket outer radius}$$

$$r_{so}(z) := \text{linterp}(\text{Skt_Out}^{\langle 0 \rangle}, \text{Skt_Out}^{\langle 1 \rangle}, z) \quad \dots \text{ socket outer radius function}$$

Bolt Radius Profiling

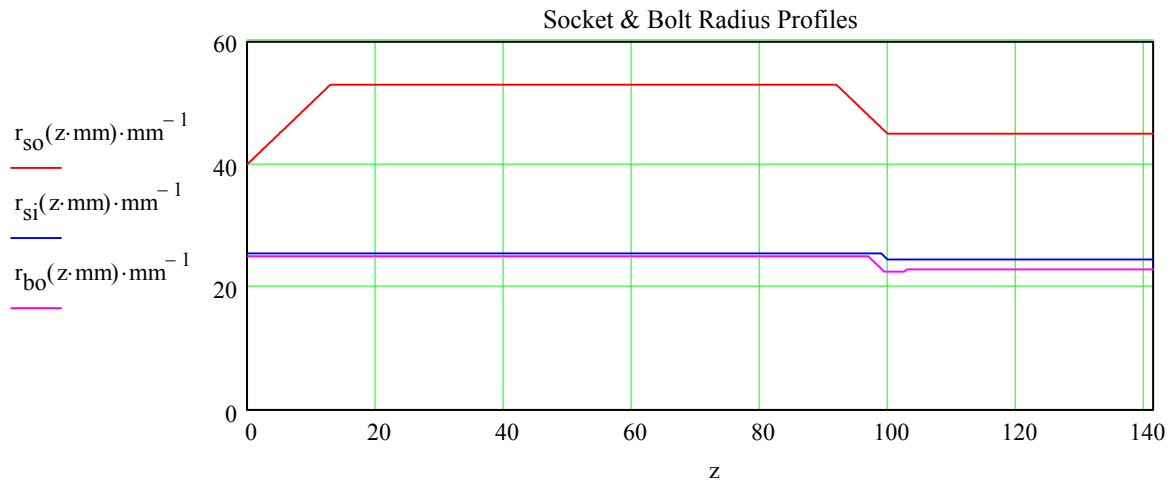
$$z_{b1} := L_{bs} + (R_b - R_{tr}) \quad z_{b1} = 99.50 \text{ mm}$$

$$z_{b2} := L_{ts} - (R_{br} - R_{tr}) \quad z_{b2} = 102.6 \text{ mm}$$

$$z_{b3} := 0.5 \cdot (L_{ts} + L_{te}) \quad z_{b3} = 141.50 \text{ mm}$$

$$\text{BlT_Out} := \begin{pmatrix} 0 \cdot \text{mm} & R_b \\ L_{bs} & R_b \\ z_{b1} & R_{tr} \\ z_{b2} & R_{tr} \\ L_{ts} & R_{br} \\ z_{b3} & R_{br} \end{pmatrix} \quad \text{BlT_Out} = \begin{matrix} z & r \text{ (mm)} \\ \begin{pmatrix} 0 & 25 \\ 97 & 25 \\ 99.5 & 22.5 \\ 102.6 & 22.5 \\ 103 & 22.9 \\ 141.5 & 22.9 \end{pmatrix} \end{matrix} \text{ mm} \quad \dots \text{ interpolating bolt outer radius}$$

$$r_{bo}(z) := \text{linterp}(\text{BlT_Out}^{\langle 0 \rangle}, \text{BlT_Out}^{\langle 1 \rangle}, z) \quad \dots \text{ bolt outer radius function}$$



Socket Modulus

$$E_s(z) := \begin{cases} E_{sb} & \text{if } z < L_{sb} \\ E_{st} & \text{otherwise} \end{cases} \quad \dots \text{bottom and top socket modulus function}$$

Socket Spring Stiffness

$$k_s := \frac{1}{\pi} \cdot \int_0^{z_{so3}} \frac{1}{E_s(z) \cdot (r_{so}(z)^2 - r_{si}(z)^2)} dz \quad k_s = 1.288 \times 10^{-7} \frac{\text{mm}}{\text{N}}$$

Equivalent Socket Modulus

$$E_{se} := \frac{z_{so3}}{k_s \cdot \pi \cdot (R_{st}^2 - R_{str}^2)} \quad E_{se} = 245.36 \text{ GPa}$$

Bolt Spring Stiffness

$$k_b := \frac{1}{\pi \cdot E_b} \cdot \int_0^{z_{so3}} \frac{1}{r_{bo}(z)^2} dz \quad k_b = 3.637 \times 10^{-7} \frac{\text{mm}}{\text{N}}$$

Equivalent Bolt Modulus

$$E_{be} := \frac{z_{so3}}{k_b \cdot \pi \cdot R_{tr}^2} \quad E_{be} = 244.61 \text{ GPa}$$

Section Axial Strain and Stress

$$\varepsilon_a(x, \varepsilon_1, \varepsilon_2) := \varepsilon_1 + \frac{1}{2} \cdot (\varepsilon_2 - \varepsilon_1) \cdot \left(1 + \frac{x}{R_{st}}\right) \quad \dots \text{function for change in axial strain}$$

$$\sigma_{ab}(x, \varepsilon_1, \varepsilon_2) := \begin{cases} E_{be} \cdot \varepsilon_a(x, \varepsilon_1, \varepsilon_2) + \frac{P_{pl}}{\pi \cdot R_{tr}^2} & \text{if } -R_{tr} \leq x \leq R_{tr} \\ 0 \cdot \text{MPa} & \text{otherwise} \end{cases} \quad \dots \text{bolt axial stress}$$

$$\sigma_{as}(x, \varepsilon_1, \varepsilon_2) := \begin{cases} \sigma_{at} \leftarrow E_{se} \cdot \varepsilon_a(x, \varepsilon_1, \varepsilon_2) - \frac{P_{pl}}{\pi \cdot (R_{st}^2 - R_{str}^2)} & \dots \text{socket axial stress} \\ \sigma_a \leftarrow \sigma_{at} & \text{if } \sigma_{at} < 0 \cdot \text{MPa} \\ \sigma_a \leftarrow 0 \cdot \text{MPa} & \text{otherwise} \\ \sigma_a & \text{if } -R_{st} \leq x \leq R_{st} \\ 0 \cdot \text{MPa} & \text{otherwise} \end{cases}$$

Cross Section Profiles

$$y_b(x) := \text{Re}\left(\sqrt{R_{tr}^2 - x^2}\right) \quad \dots \text{bolt y axis profile function}$$

$$y_{so}(x) := \text{Re}\left(\sqrt{R_{st}^2 - x^2}\right) \quad \dots \text{socket outer y axis profile function}$$

$$y_{si}(x, \varepsilon_1, \varepsilon_2) := \begin{cases} \text{Re}\left(\sqrt{R_{st}^2 - x^2}\right) & \text{if } \sigma_{as}(x, \varepsilon_1, \varepsilon_2) \geq 0 \cdot \text{MPa} \\ \text{Re}\left(\sqrt{R_{str}^2 - x^2}\right) & \text{otherwise} \end{cases} \quad \dots \text{socket inner y axis profile function}$$

Section Axial Forces

$$P_{ab}(\varepsilon_1, \varepsilon_2) := 2 \cdot \int_{-R_{st}}^{R_{st}} \int_0^{y_b(x)} \sigma_{ab}(x, \varepsilon_1, \varepsilon_2) dy dx \quad \dots \text{bolt axial force}$$

$$P_{as}(\varepsilon_1, \varepsilon_2) := 2 \cdot \int_{-R_{st}}^{R_{st}} \int_{y_{si}(x, \varepsilon_1, \varepsilon_2)}^{y_{so}(x)} \sigma_{as}(x, \varepsilon_1, \varepsilon_2) dy dx \quad \dots \text{socket axial force}$$

$$P_a(\varepsilon_1, \varepsilon_2) := P_{ab}(\varepsilon_1, \varepsilon_2) + P_{as}(\varepsilon_1, \varepsilon_2) \quad \dots \text{total axial force}$$

Section Moments

$$M_b(\varepsilon_1, \varepsilon_2) := -2 \cdot \int_{-R_{st}}^{R_{st}} \int_0^{y_b(x)} \sigma_{ab}(x, \varepsilon_1, \varepsilon_2) \cdot x dy dx \quad \dots \text{bolt moment}$$

$$M_s(\varepsilon_1, \varepsilon_2) := -2 \cdot \int_{-R_{st}}^{R_{st}} \int_{y_{si}(x, \varepsilon_1, \varepsilon_2)}^{y_{so}(x)} \sigma_{as}(x, \varepsilon_1, \varepsilon_2) \cdot x dy dx \quad \dots \text{socket moment}$$

$$M_a(\varepsilon_1, \varepsilon_2) := M_b(\varepsilon_1, \varepsilon_2) + M_s(\varepsilon_1, \varepsilon_2) \quad \dots \text{total moment}$$

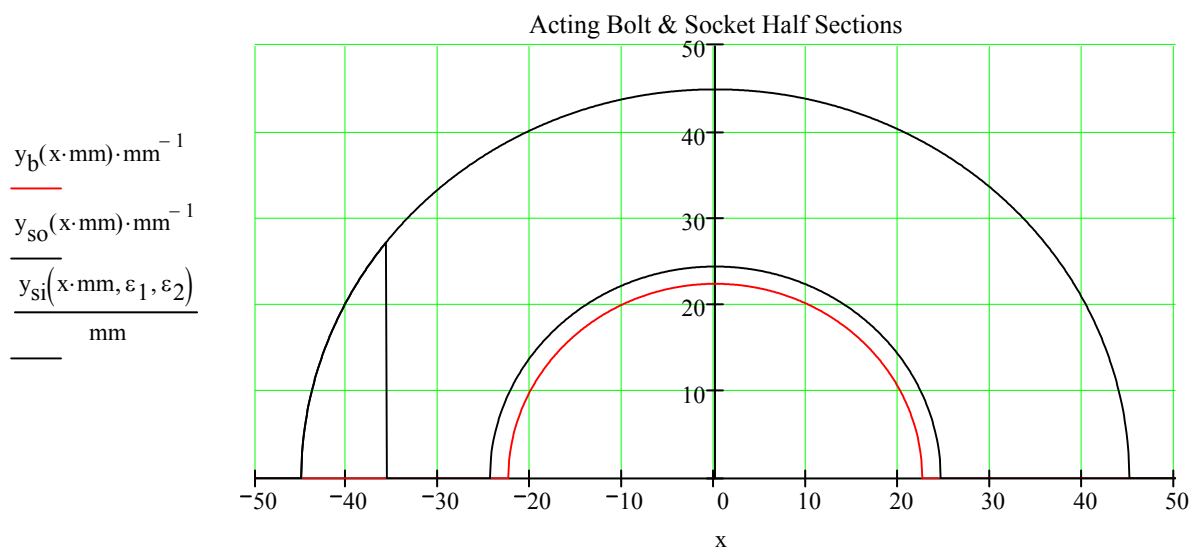
Solving for Unknown Strains

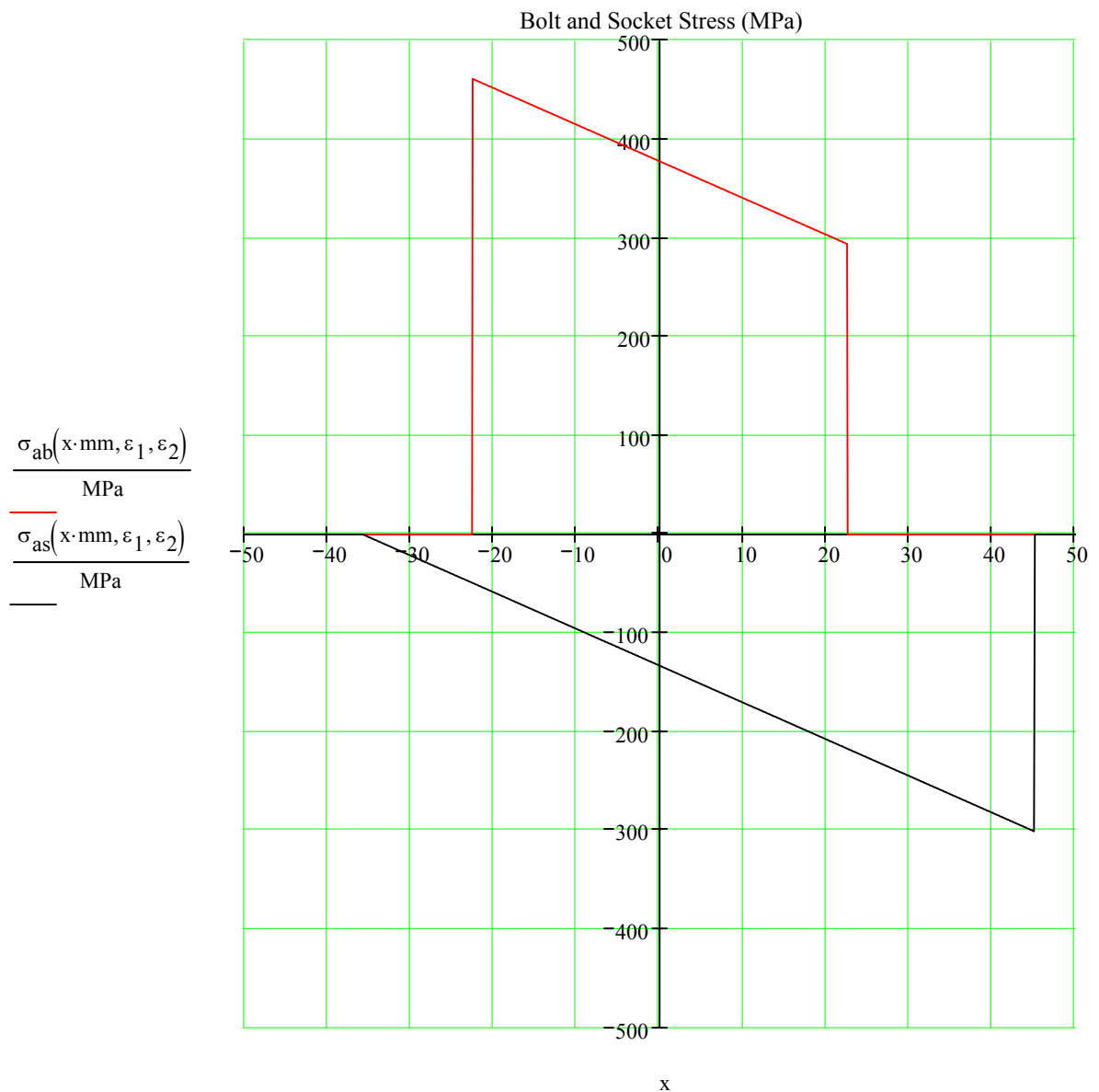
$$\text{Given} \quad P_a(\varepsilon_1, \varepsilon_2) = P_{apl} \quad \dots \text{equating to applied axial load}$$

$$M_a(\varepsilon_1, \varepsilon_2) = M_{apl} \quad \dots \text{equating to applied BM}$$

$$\text{Giving ...} \quad \varepsilon_1 = 6.868 \times 10^{-4} \quad \dots \text{and ...} \quad \varepsilon_2 = -6.804 \times 10^{-4}$$

Checks ...	$P_{ab}(\varepsilon_1, \varepsilon_2) = 601253.7 \text{ N}$	... bolt axial load	
	$P_{as}(\varepsilon_1, \varepsilon_2) = -601253.7 \text{ N}$	... socket axial load	
	$P_a(\varepsilon_1, \varepsilon_2) = -0 \text{ N}$	... total internal axial load, where ...	$P_{apl} = 0 \text{ N}$
	$M_b(\varepsilon_1, \varepsilon_2) = 747.9 \text{ N}\cdot\text{m}$	... bolt moment	
	$M_s(\varepsilon_1, \varepsilon_2) = 10752.1 \text{ N}\cdot\text{m}$	... socket moment	
	$M_a(\varepsilon_1, \varepsilon_2) = 11500.0 \text{ N}\cdot\text{m}$	... total internal moment, where ...	$M_{apl} = 11500 \text{ N}\cdot\text{m}$
	$\frac{M_b(\varepsilon_1, \varepsilon_2)}{M_{apl}} = 6.50 \%$	... percentage of BM seen by bolt	
	$\frac{M_a(\varepsilon_1, \varepsilon_2)}{M_{apl}} = 100.00 \%$	... internal BM to applied	





Bolt stresses ...	$\sigma_{ab}(-R_{tr}, \varepsilon_1, \varepsilon_2) = 461.6 \text{ MPa}$	... thread undercut, LHS
	$\sigma_{ab}(R_{tr}, \varepsilon_1, \varepsilon_2) = 294.4 \text{ MPa}$	... thread undercut, RHS
Socket stress ...	$\sigma_{as}(-R_{st}, \varepsilon_1, \varepsilon_2) = 0.0 \text{ MPa}$	... top socket outer radius, LHS
	$\sigma_{as}(R_{st}, \varepsilon_1, \varepsilon_2) = -301.0 \text{ MPa}$	... top socket outer radius, RHS