

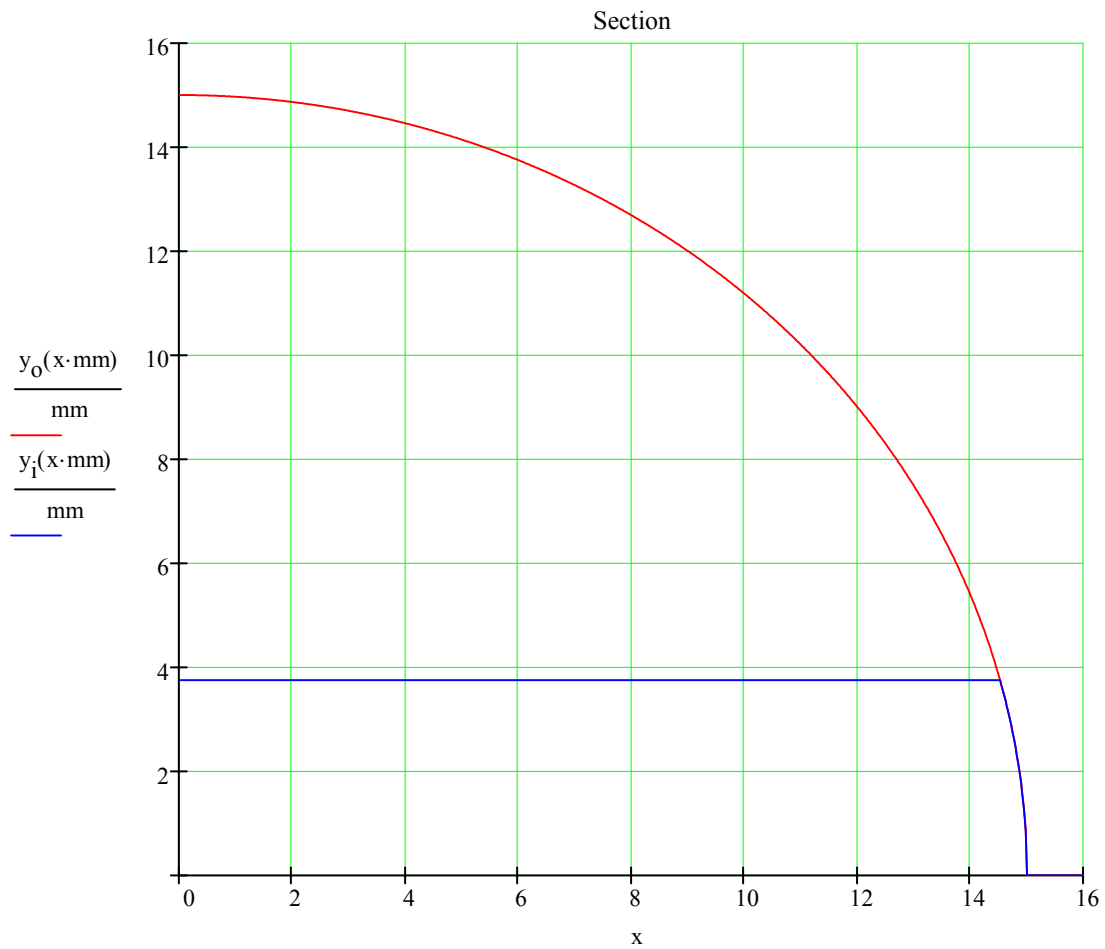
$R := 15 \cdot \text{mm}$... outer shaft radius

$w := 7.5 \cdot \text{mm}$... slot width

$y_o(x) := \text{Re}\left(\sqrt{R^2 - x^2}\right)$... outer radius function

$y_i(x) := \min\left(\frac{w}{2}, y_o(x)\right)$... slot function

$L^2 + \left(\frac{t}{2}\right)^2 = R^2 \quad L := \sqrt{R^2 - \left(\frac{w}{2}\right)^2} \quad L = 14.524 \text{ mm} \quad \dots \text{ intersection distance}$



$$\frac{(x-a)}{(L-a)} \quad \dots \text{linearly increasing component, for contact loading}$$

$$\frac{\left(2 \cdot \sqrt{R^2 - x^2} - w\right)}{(2 \cdot R - w)} \quad \dots \text{thickness factor, 1 at } x=0 \text{ and 0 at } x=L$$

$$\cos \left[\frac{\pi}{2} \cdot \frac{(2 \cdot y - w)}{(2 \cdot R - w)} \right] \quad \dots \text{change in shear stress direction through thickness}$$

$$\tau(x, y, a, k_s) := k_s \cdot \frac{(x-a)}{(L-a)} \cdot \frac{\left(2 \cdot \sqrt{R^2 - x^2} - w\right)}{(2 \cdot R - w)} \cdot \cos \left[\frac{\pi}{2} \cdot \frac{(2 \cdot y - w)}{(2 \cdot R - w)} \right] \cdot (x \geq a) \quad \dots \text{shear stress function}$$

$$dP = \tau \cdot dA \quad \dots \text{elemental shear force}$$

$$dT = x \cdot dP \quad \dots \text{elemental torque}$$

$$dT = k_s \cdot x \cdot \frac{(x-a)}{(L-a)} \cdot \frac{\left(2 \cdot \sqrt{R^2 - x^2} - w\right)}{(2 \cdot R - w)} \cdot \cos \left[\frac{\pi}{2} \cdot \frac{(2 \cdot y - w)}{(2 \cdot R - w)} \right] \cdot dA \quad \dots \text{substituting}$$

Integrating over section area and equating to half of the applied torque T_a

$$\frac{T_a}{2} = \frac{k_s}{(2 \cdot R - w) \cdot (L - a)} \cdot \int_a^L \int_{0.5 \cdot w}^{\sqrt{R^2 - x^2}} x \cdot (x - a) \cdot \left(2 \cdot \sqrt{R^2 - x^2} - w\right) \cdot \cos \left[\frac{\pi}{2} \cdot \frac{(2 \cdot y - w)}{(2 \cdot R - w)} \right] dy dx$$

$$T_a := 200 \cdot \text{N} \cdot \text{m} \quad \dots \text{applied torque}$$

$$\text{MPa} := 10^6 \cdot \text{Pa} \quad \dots \text{setting stress units}$$

$$k_s(a) := \frac{T_a \cdot (2 \cdot R - w) \cdot (L - a)}{2 \cdot \int_a^L \int_{0.5 \cdot w}^{\sqrt{R^2 - x^2}} x \cdot (x - a) \cdot \left(2 \cdot \sqrt{R^2 - x^2} - w\right) \cdot \cos \left[\frac{\pi}{2} \cdot \frac{(2 \cdot y - w)}{(2 \cdot R - w)} \right] dy dx} \quad \dots \text{rearranged for } k_s$$

When start of contact is at middle of contact line, i.e. $x = 0$

$$ks_0 := ks(0 \cdot \text{mm}) \quad ks_0 = 487.1 \text{ MPa}$$

$$Ps_0 := \int_0^L \int_{0.5 \cdot w}^{\sqrt{R^2 - x^2}} \tau(x, y, 0 \text{ mm}, ks_0) dy dx \quad Ps_0 = 13061 \text{ N} \quad \frac{Ta}{2 \cdot Ps_0} = 7.656 \text{ mm}$$

$$\tau_0(x) := \tau(x, 0 \cdot \text{mm}, 0 \cdot \text{mm}, ks_0) \quad \dots \text{ shear stress function}$$

When start of contact is half way along contact line, i.e. $x = L/2$

$$ks_L := ks(0.5 \cdot L) \quad ks_L = 1237.7 \text{ MPa}$$

$$Ps_L := \int_0^L \int_{0.5 \cdot w}^{\sqrt{R^2 - x^2}} \tau(x, y, 0.5 \cdot L, ks_L) dy dx \quad Ps_L = 9261 \text{ N} \quad \frac{Ta}{2 \cdot Ps_L} = 10.798 \text{ mm}$$

$$\tau_L(x) := \tau(x, 0 \cdot \text{mm}, 0.5 \cdot L, ks_L) \quad \dots \text{ shear stress function}$$

