

$f_{m_1} := 4.22\text{Hz}$       Egensvining fra modeshape 1       $f_{w_1} := 1.55\text{Hz}$       bevægelsesfrekvensen for personen  
 $f_{m_2} := 6.59\text{Hz}$       Egensvining fra modeshape 2       $h_1 := 1$        $h_2 := 2$        $h_3 := 3$        $h_4 := 4$       harmonisk lastkomponent  
 $f_{m_3} := 16.88\text{Hz}$       Egensvining fra modeshape 3

$f := 1.55 \cdot 2$   
 $0.41 \cdot (f - 0.95) = 0.882$

$F_h := 700\text{N} \cdot [0.41 \cdot (f - 0.95)] = 617.05 \cdot \text{N}$       gennemsnitlige vægt af person

$\zeta_m := 1.5\%$   
 $m_m := 36960\text{kg} = 3.696 \times 10^4 \cdot \text{kg}$  modal massen

$f_{h_1} := h_1 \cdot f_{w_1} = 1.55 \cdot \text{Hz}$       Harmonisk lastfrekvens fra den 1.harmoniske lastkomponent og bevægelsesfrekvens  $f_{w_1}$   
 $f_{h_2} := h_2 \cdot f_{w_1} = 3.1 \cdot \text{Hz}$       Harmonisk lastfrekvens fra den 2.harmoniske lastkomponent og bevægelsesfrekvens  $f_{w_1}$   
 $f_{h_3} := h_3 \cdot f_{w_1} = 4.65 \cdot \text{Hz}$       Harmonisk lastfrekvens fra den 3. harmoniske lastkomponent og bevægelsesfrekvens  $f_{w_1}$   
 $f_{h_4} := h_4 \cdot f_{w_1} = 6.2 \cdot \text{Hz}$       Harmonisk lastfrekvens fra den 4. harmoniske lastkomponent og bevægelsesfrekvens  $f_{w_1}$

$q := 1848 \frac{\text{kg}}{\text{m}} \cdot 10 \frac{\text{m}}{\text{s}^2}$        $L := 20\text{m}$   
 $E := 38\text{GPa}$   
 $I := 0.056\text{m}^4$

$\mu_{t_m_1} := \frac{5}{384} \cdot \frac{q \cdot L^4}{E \cdot I} = 18.09210526 \cdot \text{mm}$        $5.74 \cdot 10^{(-4)} = 0.000574$

$Q := 36960\text{kg} \cdot 10 \frac{\text{m}}{\text{s}^2}$

$y := \frac{20\text{m}}{2}$

$\mu_{e_m_1} := \sin\left(\frac{\pi \cdot y}{L}\right) = 1$

$\rho_{h_m} := 1$

For a particular footfall rate,  $f_w$ , it is required to calculate the response of each of the first four harmonics:

Firstly, for each harmonic  $h$ , from  $h = 1$  to  $h = 4$ .

1. Calculate the harmonic forcing frequency,  $f_h$ :

$f_h = h f_w$

2. Calculate the harmonic force,  $F_h$ , at this harmonic frequency

For each mode,  $m$

3. Calculate the real and imaginary acceleration ( $a_{real,h,m}$ ,  $a_{imag,h,m}$ )

$F_h = DLF \cdot P$

where  $DLF$  is calculated from Table 4.3 (Design value) and  $P$  is the weight of the walker.

$a_{real,h,m} = \left(\frac{f_h}{f_m}\right)^2 \frac{F_h \mu_{tm} \mu_{em} \rho_{hm}}{r \tilde{m}_m} \frac{A_m}{A_m^2 + B_m^2}$

$a_{imag,h,m} = \left(\frac{f_h}{f_m}\right)^2 \frac{F_h \mu_{tm} \mu_{em} \rho_{hm}}{r \tilde{m}_m} \frac{B_m}{A_m^2 + B_m^2}$

$$A_m := 1 - \left( \frac{f_{h1}}{f_{m1}} \right)^2 = 0.865$$

$$B_m := 2 \cdot \zeta_m \cdot \frac{f_{h1}}{f_{m1}} = 0.011$$

$$\alpha_{real\_h\_m\_1} := \left( \frac{f_{h1}}{f_{m1}} \right)^2 \cdot \frac{F_{h1} \cdot \mu_{t\_m\_1} \cdot \mu_{e\_m\_1} \cdot \rho_{h\_m}}{m_m} \cdot \frac{A_m}{(A_m)^2 + (B_m)^2} = 0.0000470959 \frac{m}{s^2} \quad \text{real acceleration ved modeshape 1}$$

$$\alpha_{imag\_h\_m\_1} := \left( \frac{f_{h1}}{f_{m1}} \right)^2 \cdot \frac{F_{h1} \cdot \mu_{t\_m\_1} \cdot \mu_{e\_m\_1} \cdot \rho_{h\_m}}{m_m} \cdot \frac{B_m}{(A_m)^2 + (B_m)^2} = 0.0000005999 \frac{m}{s^2}$$

where  $A_m = 1 - \left( \frac{f_h}{f_m} \right)^2$ ,  $B_m = 2\zeta_m \frac{f_h}{f_m}$ ,  $\rho_{h,m}$  is from

$\zeta_m$  can be estimated from Table A2 and  $f_m$ ,  $\hat{m}_m$  and  $\mu$  are found described in Appendix A.

4. Sum the real and imaginary responses in all modes to yield the real and imaginary acceleration to this harmonic force,  $a_h$ :

$$a_{real,h} = \sum_m a_{real,h,m}; \quad a_{imag,h} = \sum_m a_{imag,h,m}$$

5. Find the magnitude of this acceleration  $|a_h|$  which is the total acceleration to this harmonic (at this frequency):

$$|a_h| = \sqrt{a_{real,h}^2 + a_{imag,h}^2}$$

in each mode to

(4.3)

from Table 4.3:

) in each mode:

equals the static

(4.4)

equation 4.2

and using methods

the total real and

(4.5)

response in all

(4.6)